# Nonlinear Finite Element Methods Material nonlinearities

Vinh Phu Nguyen

# Sources of nonlinearities

Geometrical nonlinearities

large displacement/rotation (structural instability)

finite deformation: large strain (metal forming)

Material nonlinearities

plasticity

cohesive zone models

damage

visco-plasticity

### Boundary nonlinearities contact problems 2

$$\boldsymbol{\sigma} = f(\boldsymbol{\epsilon}, \boldsymbol{\alpha})$$

# Sources of nonlinearities

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## References

Nonlinear finite elements for continua and structures, T. Belytschko, W.K. Liu and B. Moran, Wiley, 2000.

Nonlinear finite element methods, P. Wriggers, Springer, 2008.

Non-linear finite element analysis of solids and structures, R. de Borst, M.A. Crisfield, J.J.C. Remmers, C.V. Verhoosel, Wiley, 2012.

Nonlinear continuum mechanics for finite element analysis, J. Bonet and R.D. Wood, Cambridge, 1997.



Thermodynamics+experiments

Rate independent

Rate dependent

(Time independent)

Elastic

(Time dependent)

strain rate Viscoelastic

Elasto-plastic

Viscoplastic

Elasto-damage

Visco-elasto-damage

Plastic-damage

$$\int_{\Omega} \delta \boldsymbol{\epsilon}^{\mathrm{T}} \boldsymbol{\sigma} \mathrm{d}\Omega = \int_{\Gamma_t} \delta \mathbf{u}^{\mathrm{T}} \bar{\mathbf{t}} \mathrm{d}\Gamma_t$$

#### Linear problems

$$\int_{\Omega} \delta \boldsymbol{\epsilon}^{\mathrm{T}} \mathbf{D}^{e} \boldsymbol{\epsilon} \mathrm{d}\Omega = \int_{\Gamma_{t}} \delta \mathbf{a}^{\mathrm{T}} \bar{\mathbf{t}} \mathrm{d}\Gamma_{t}$$

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#### Linear problems

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#### Linear problems

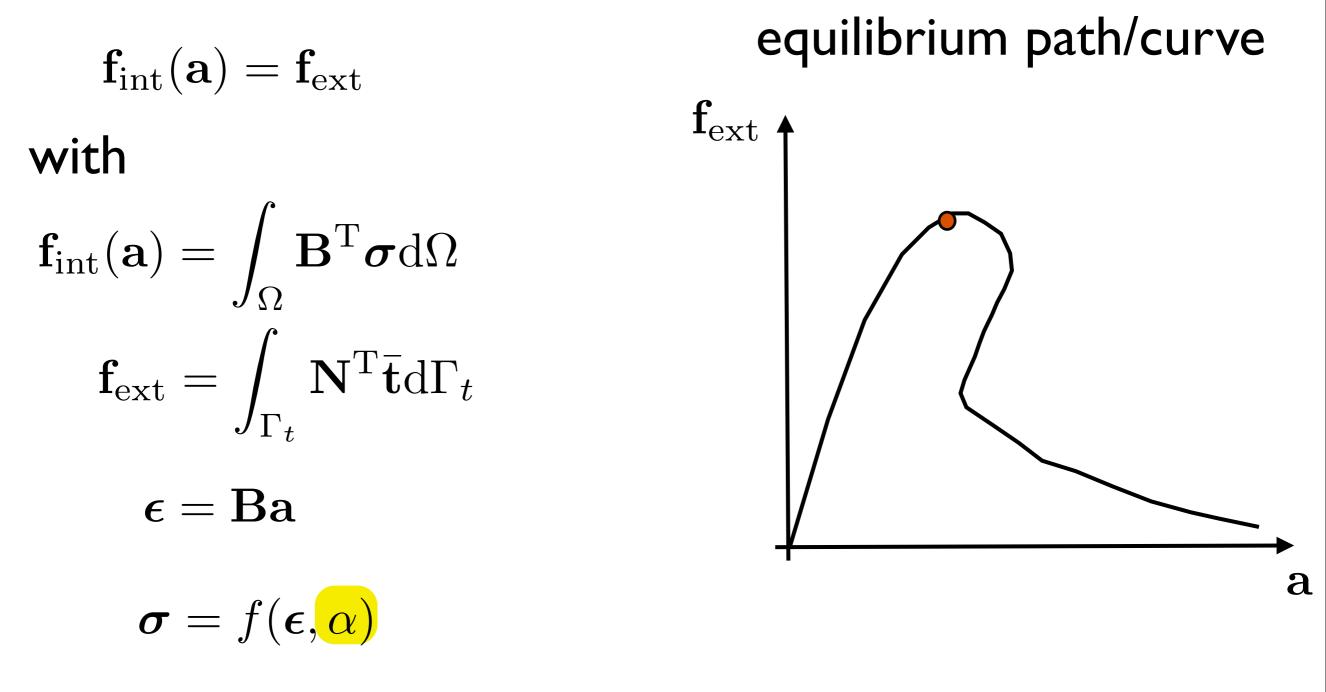
Nonlinear problems

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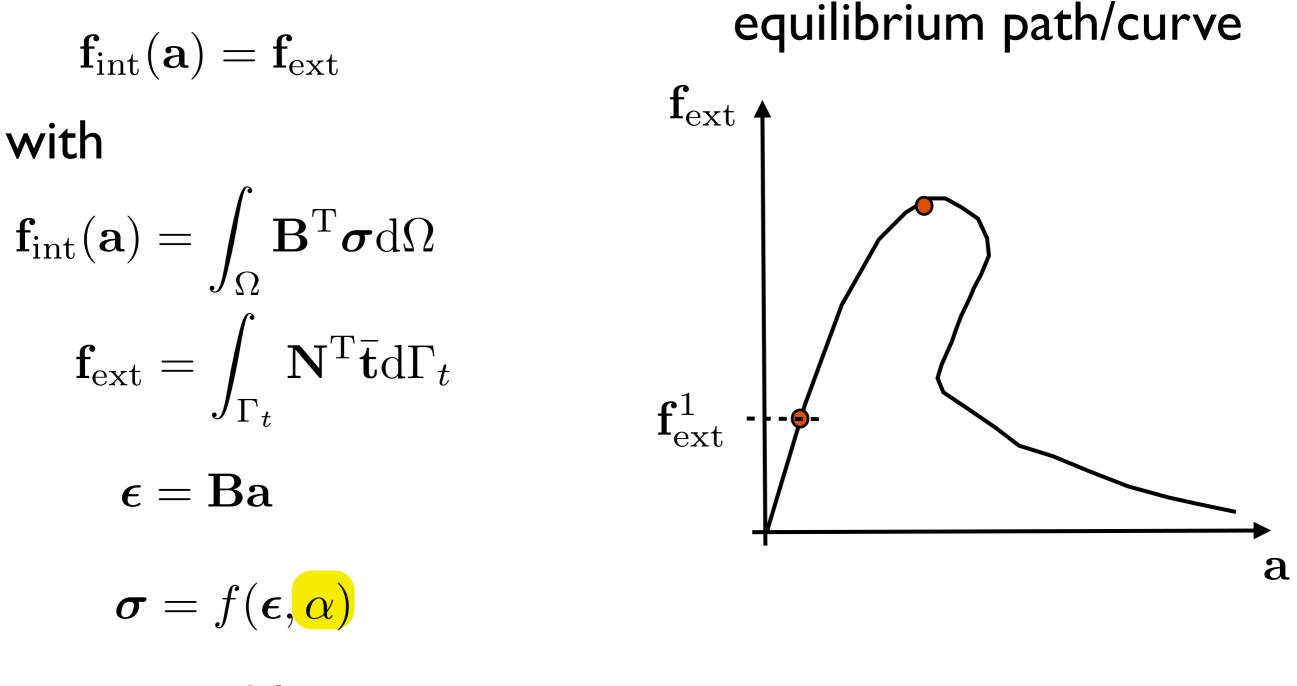
### Discrete equilibrium equation



### history variables

Argyris (1964)

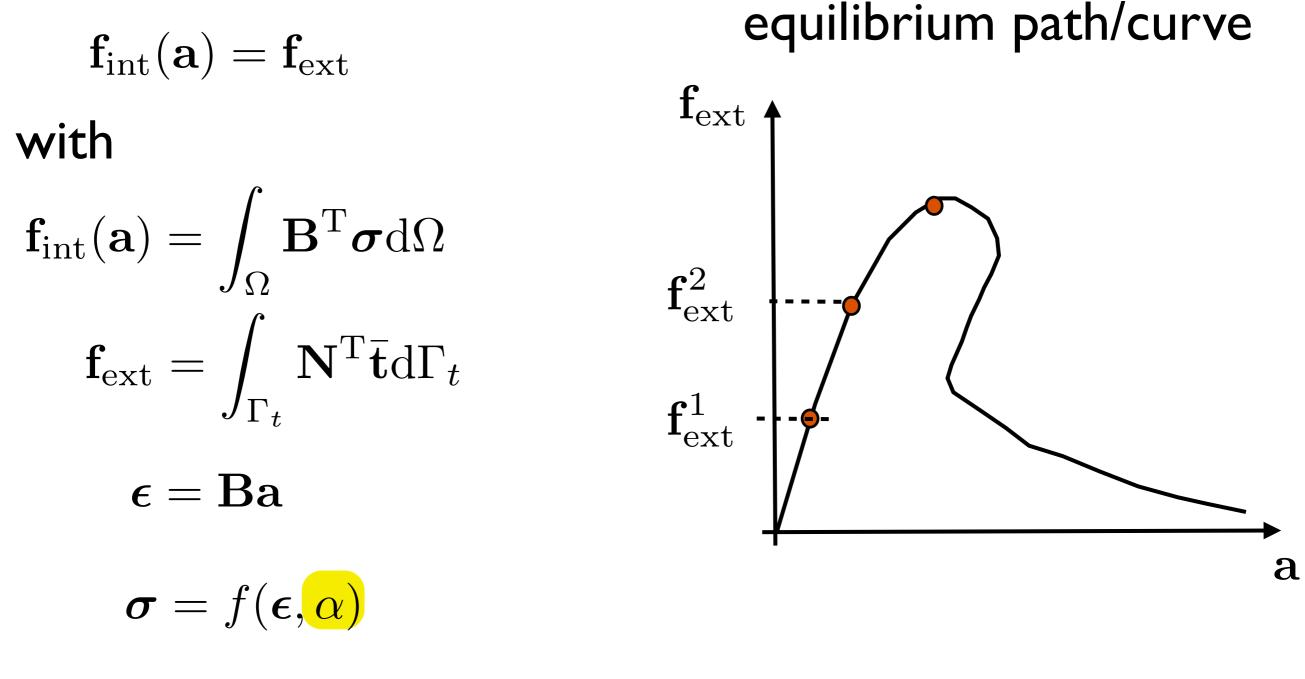
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history variables

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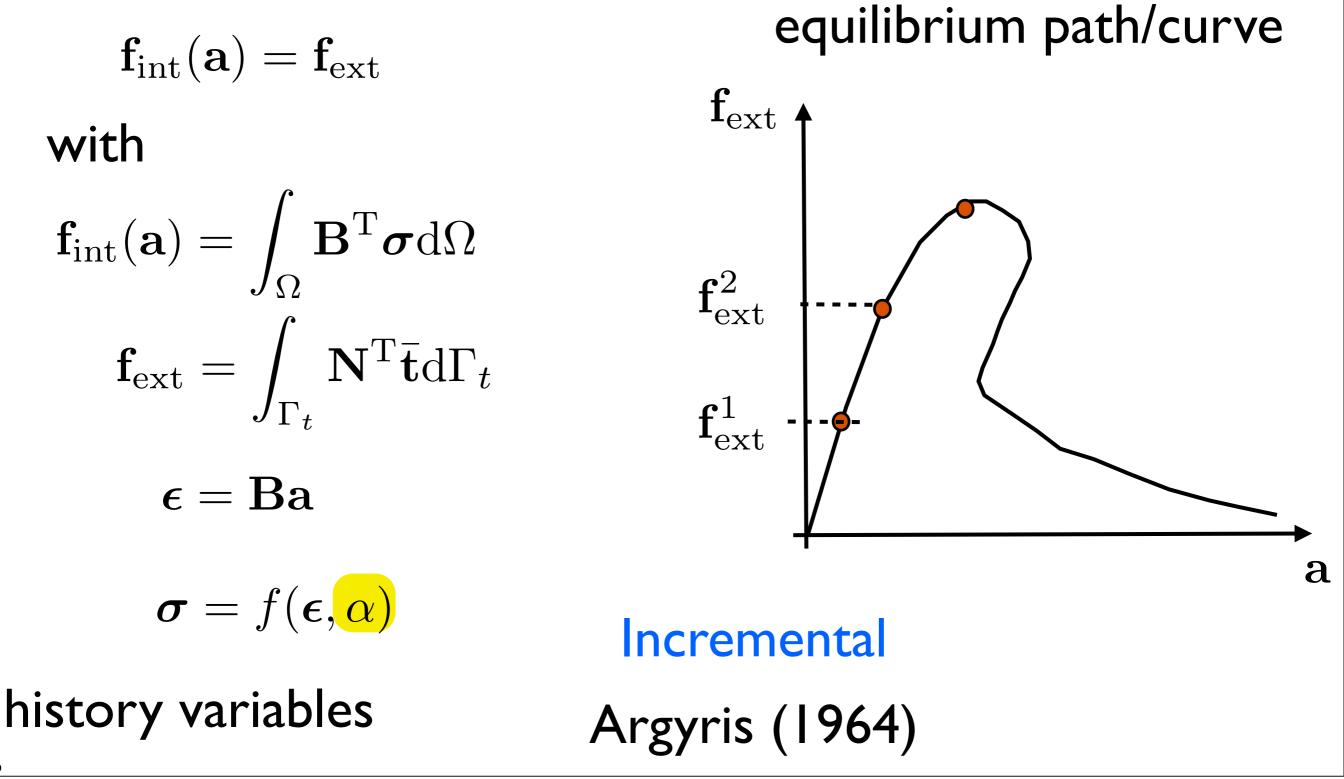
### Discrete equilibrium equation



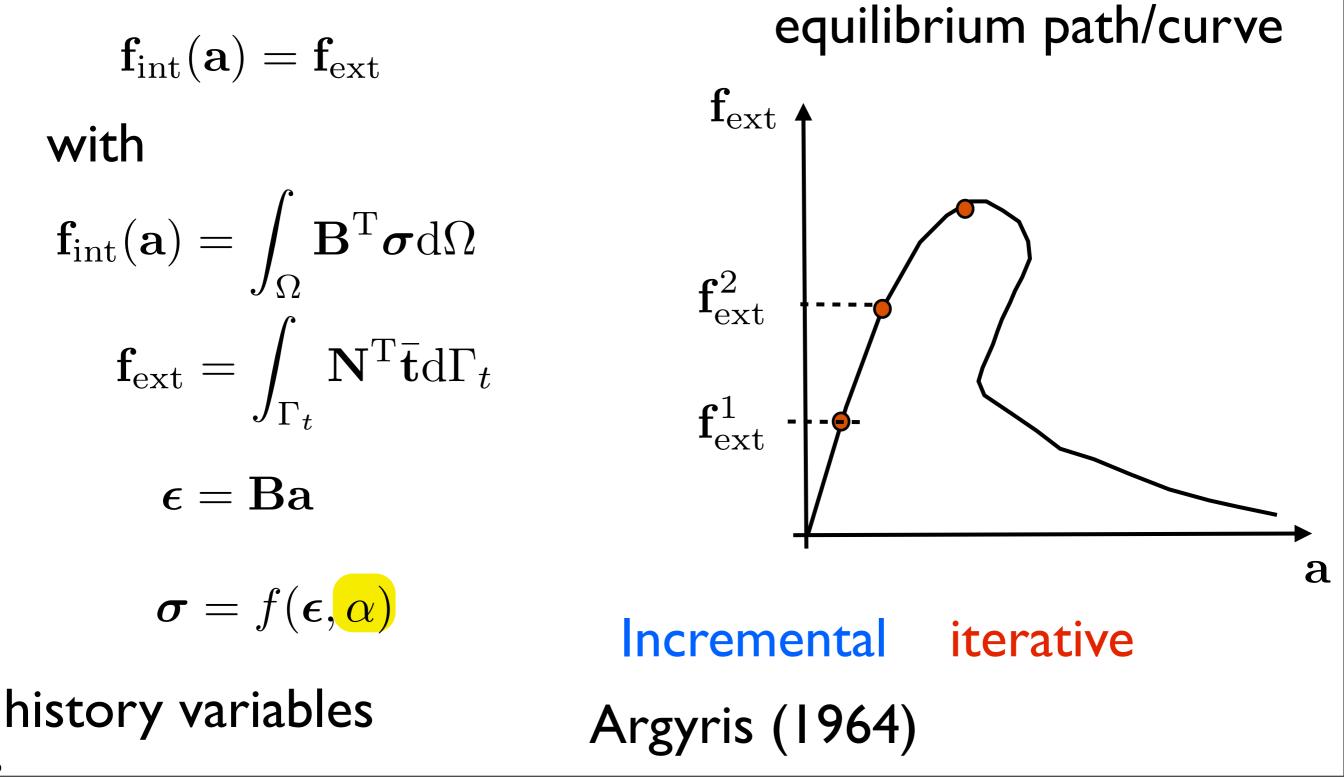
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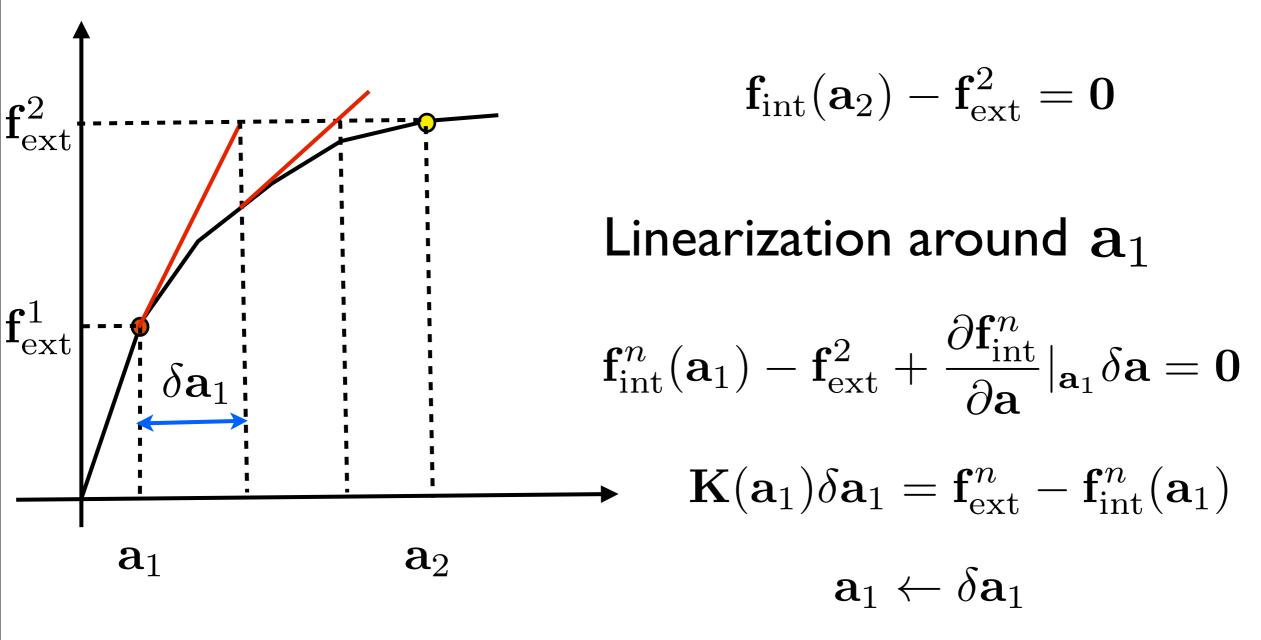
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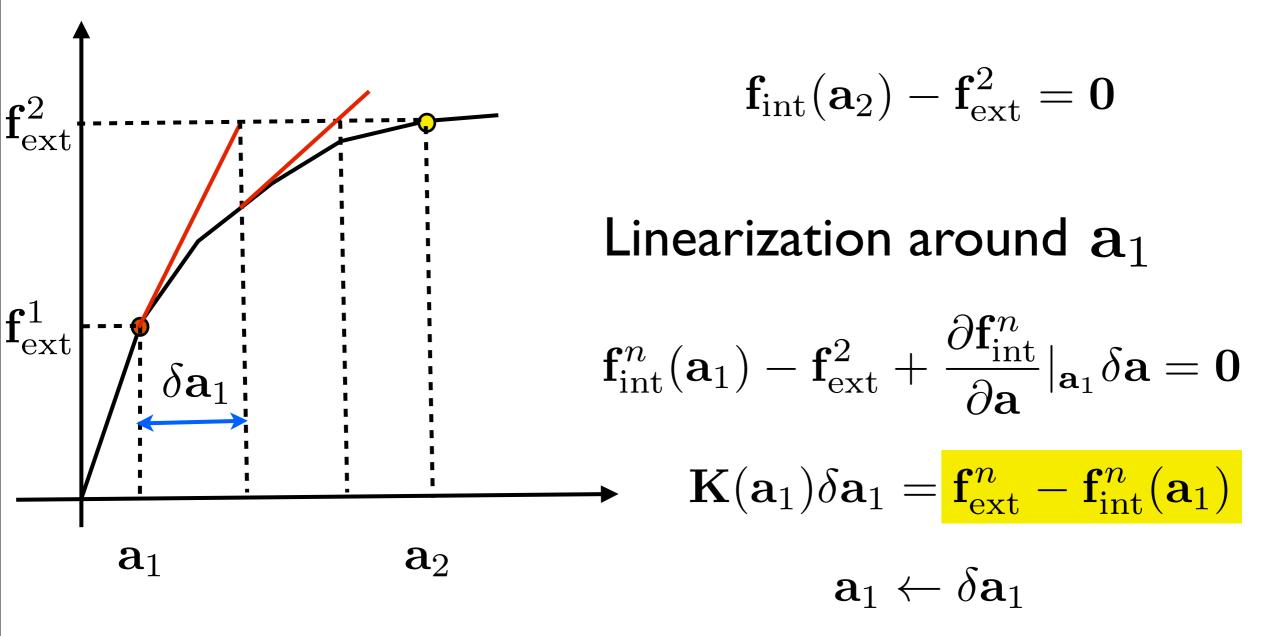


Full NR scheme



Tangent stiffness matrix  $\begin{bmatrix} \mathbf{K} = \frac{\partial \mathbf{f}_{int}}{\partial \mathbf{a}} \end{bmatrix}$ 

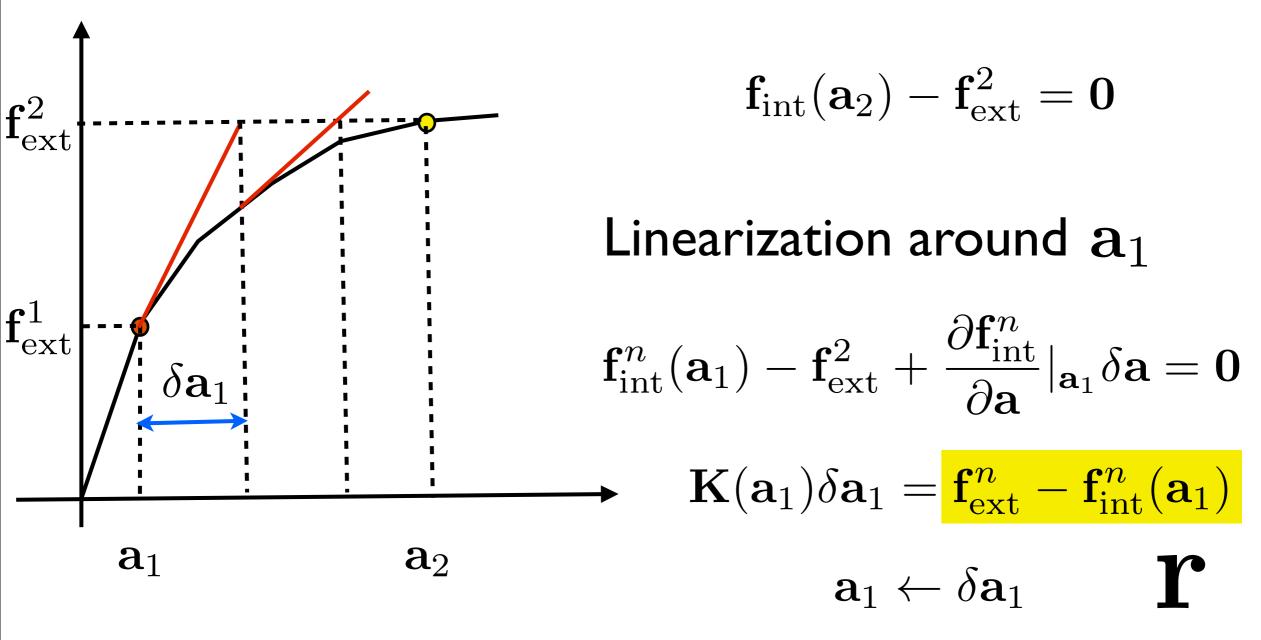
Full NR scheme



Tangent stiffness matrix

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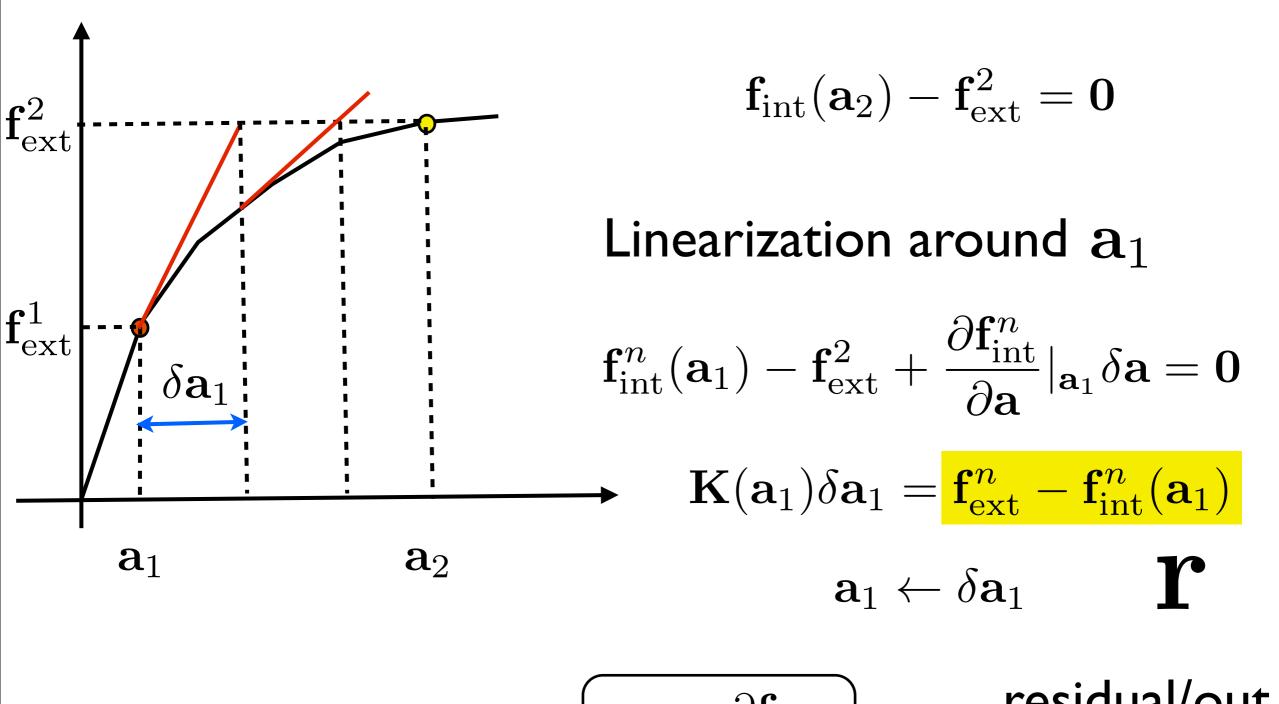
Full NR scheme



Tangent stiffness matrix

$$\mathbf{K} = \frac{\partial \mathbf{f}_{\text{int}}}{\partial \mathbf{a}}$$

Full NR scheme



Tangent stiffness matrix

$$\mathbf{K} = \frac{\partial \mathbf{f}_{\text{int}}}{\partial \mathbf{a}}$$

residual/out of balance vector

## Tangent stiffness matrix

$$\mathbf{f}_{\text{int}}(\mathbf{a}) = \int_{\Omega} \mathbf{B}^{\mathrm{T}} \boldsymbol{\sigma} \mathrm{d}\Omega$$
$$\mathbf{f}_{\text{int}}(\mathbf{a}) = \int_{\Omega} \mathbf{B}^{\mathrm{T}} \boldsymbol{\sigma}[\boldsymbol{\epsilon}(\mathbf{a})] \mathrm{d}\Omega$$

$$\mathbf{K} = \int_{\Omega} \mathbf{B}^{\mathrm{T}} \frac{\partial \boldsymbol{\sigma}[\boldsymbol{\epsilon}(\mathbf{a})]}{\partial \mathbf{a}} \mathrm{d}\Omega$$

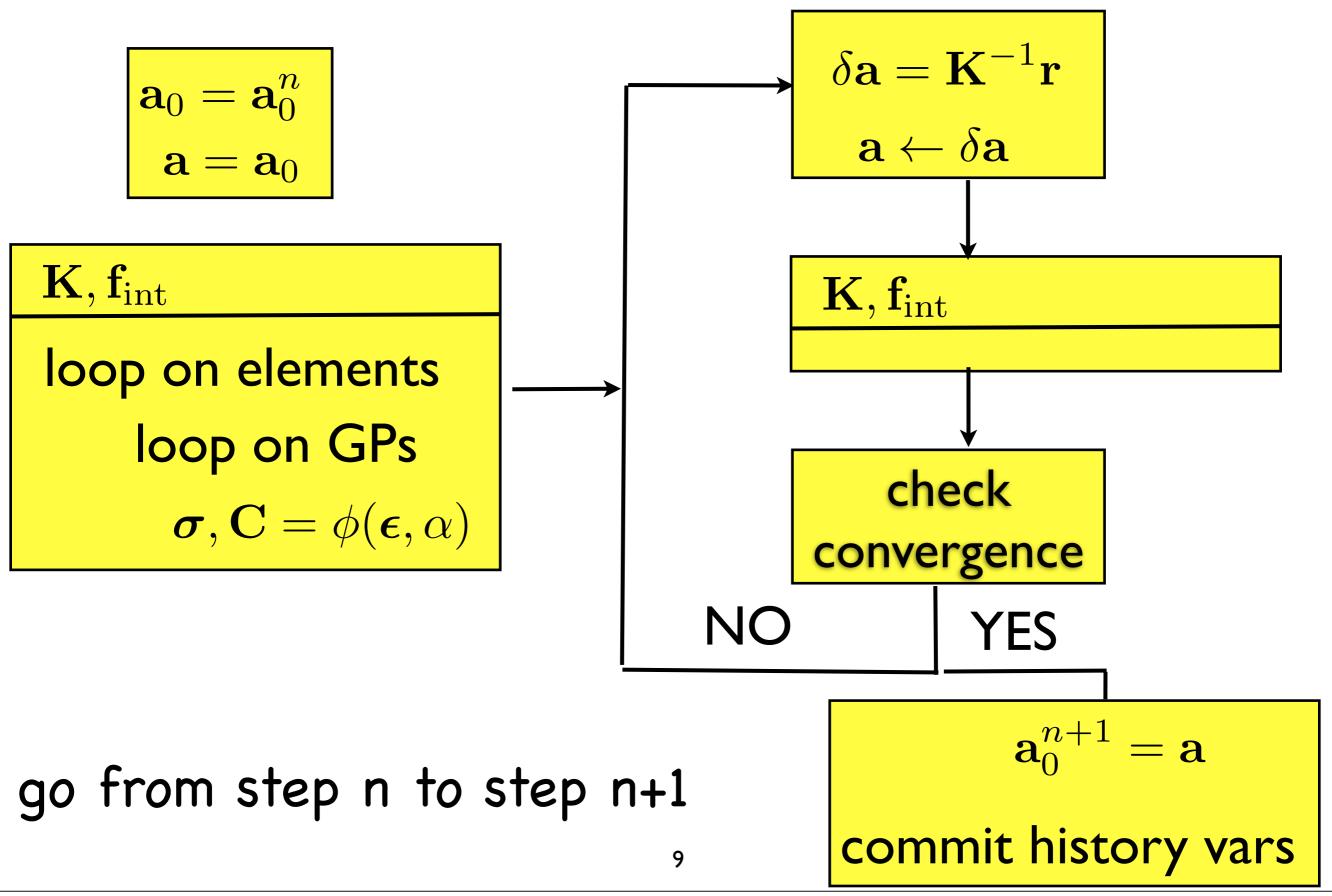
$$\mathbf{K} = \int_{\Omega} \mathbf{B}^{\mathrm{T}} \frac{\partial \boldsymbol{\sigma}}{\partial \boldsymbol{\epsilon}} \mathbf{B} \mathrm{d}\Omega$$

$$\mathbf{K} = \frac{\partial \mathbf{f}_{\text{int}}^n}{\partial \mathbf{a}}$$

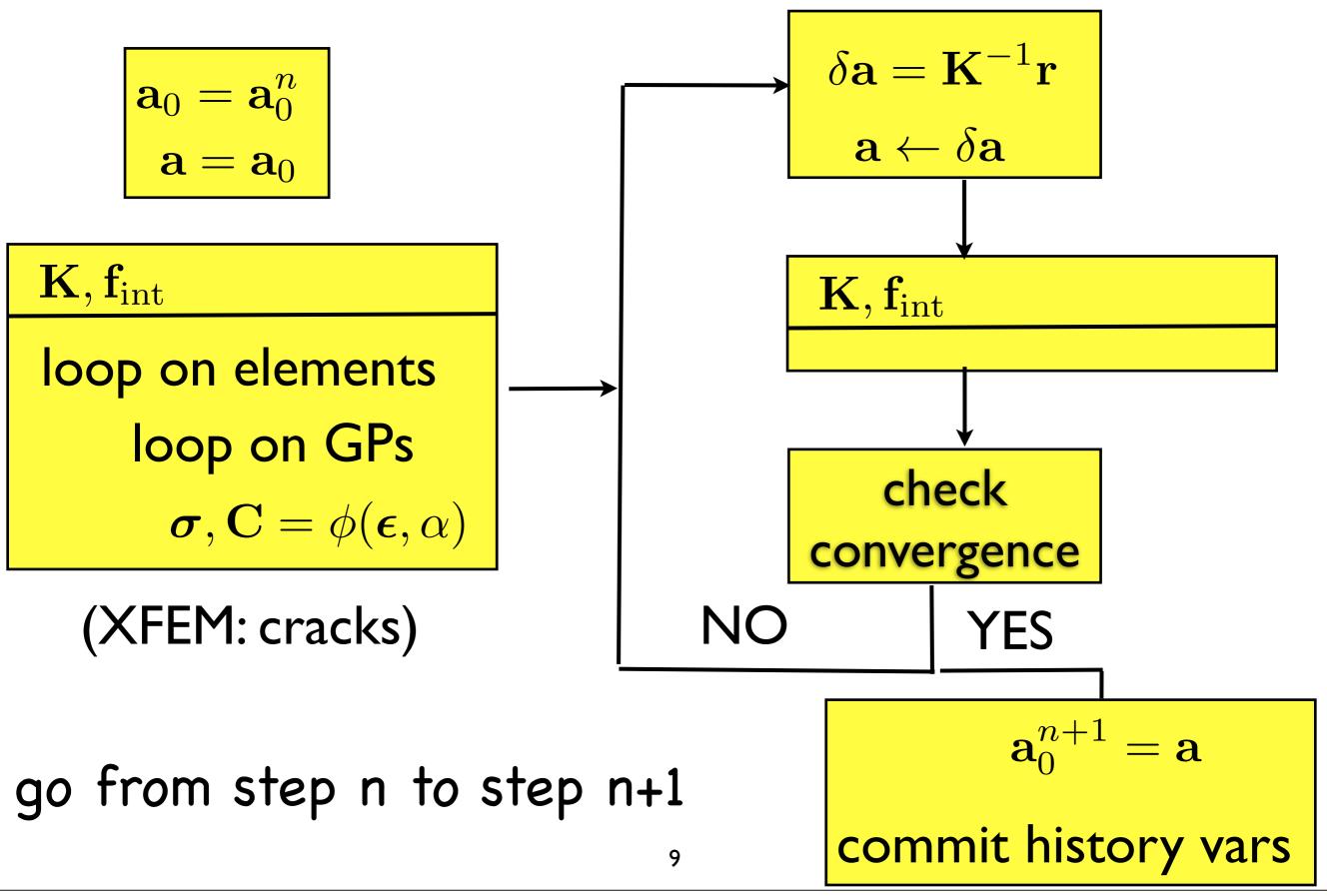
$$d\boldsymbol{\sigma} = \mathbf{C}d\boldsymbol{\epsilon}$$

$$\mathbf{\mathbf{K}} = \int_{\Omega} \mathbf{B}^{\mathrm{T}} \mathbf{C} \mathbf{B} \mathrm{d} \Omega$$

### Flowchart



### Flowchart



## At integration level

For one Gauss point, do $\mathbf{a} = \mathbf{N}\mathbf{a}_e$ I. compute displacement  $\mathbf{a}$  $\boldsymbol{\epsilon} = \mathbf{B}\mathbf{a}$ 2. compute strains  $\boldsymbol{\epsilon}$ , get history  $\boldsymbol{\alpha}$  $\boldsymbol{\sigma} = f(\boldsymbol{\epsilon}, \boldsymbol{\alpha})$ 3. compute stresses/material tangent $\mathbf{f}_{int} = \int_{\Omega} \mathbf{B}^T \boldsymbol{\sigma} d\Omega$ 4. compute internal force vector $\mathbf{K} \leftarrow \int_{\Omega} \mathbf{B}^T \mathbf{C} \mathbf{B} d\Omega$ 

## At integration level

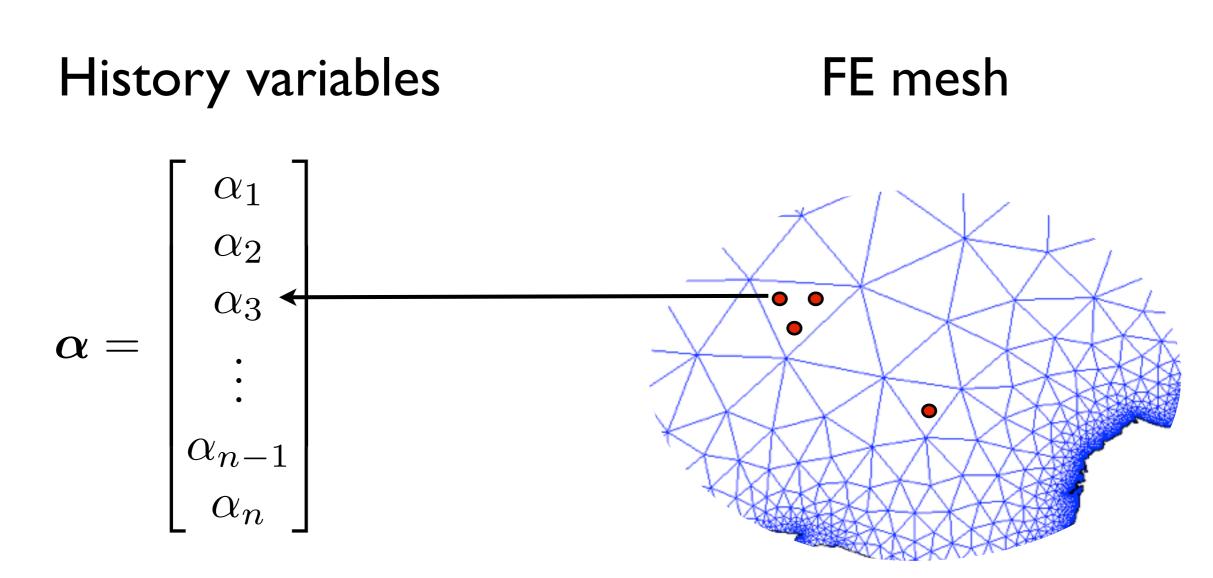
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### stress update

# Storage of history vars



 $\alpha_0$ : history vars at previous converged load step n: number of integration points of the mesh

# Isotropic damage model

$$\vec{\sigma} = (1 - \omega)\mathbf{D}\epsilon$$

$$\kappa = \max \epsilon_{eq}$$

$$\omega = f(\kappa)$$

$$\epsilon_{eq} = g(\epsilon)$$

$$\vec{\sigma} = (1 - \omega)\mathbf{D}\dot{\epsilon} - \mathbf{D}\epsilon\dot{\omega}$$

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$$\vec{\sigma} = (1 - \omega)\mathbf{D}\dot{\epsilon} - \mathbf{D}\epsilon\frac{\partial f}{\partial\epsilon_{eq}}\frac{\partial\epsilon_{eq}}{\partial\epsilon}\dot{\epsilon}$$

$$\vec{\sigma} = \begin{bmatrix} (1 - \omega)\mathbf{D} - \mathbf{D}\epsilon\frac{\partial f}{\partial\epsilon_{eq}}\frac{\partial\epsilon_{eq}}{\partial\epsilon}\end{bmatrix}\dot{\epsilon}$$

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$$\vec{c}$$

$$\mathbf{C}$$

$$\mathbf{C}$$

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$$\omega = 1 - \frac{\kappa}{\kappa_I} [1 - \alpha + \alpha \exp^{-\beta(\kappa - \kappa_I)}], \quad \kappa \ge \kappa_I$$

$$\epsilon_{\rm eq} = \sqrt{\langle \epsilon_1 \rangle^2 + \langle \epsilon_2 \rangle^2 + \langle \epsilon_3 \rangle^2} \qquad \langle x \rangle = 0.5(x + |x|)$$

$$\frac{\partial \epsilon_{\text{eq}}}{\partial \epsilon_{ij}} = \frac{1}{\epsilon_{\text{eq}}} \left( \langle \epsilon_1 \rangle \frac{\partial \epsilon_1}{\partial \epsilon_{ij}} + \langle \epsilon_2 \rangle \frac{\partial \epsilon_2}{\partial \epsilon_{ij}} + \langle \epsilon_3 \rangle \frac{\partial \epsilon_3}{\partial \epsilon_{ij}} \right)$$

$$\langle \epsilon_k \rangle \frac{\partial \langle \epsilon_k \rangle}{\partial \epsilon_{ij}} = \begin{cases} \epsilon_k \frac{\partial \epsilon_k}{\partial \epsilon_{ij}} & \text{if } \epsilon_k > 0\\ 0 & \text{if } \epsilon_k \le 0 \end{cases}$$

$$\epsilon^{3} - I_{1}\epsilon^{2} + I_{2}\epsilon - I_{3} = 0 \qquad I_{1} = \epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz}$$

$$I_{1} = \epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz}$$

$$I_{2} = \epsilon_{xx}\epsilon_{yy} + \epsilon_{yy}\epsilon_{zz} + \epsilon_{zz}\epsilon_{xx} - \epsilon_{xy}^{2} - \epsilon_{yz}^{2} - \epsilon_{zx}^{2}$$

$$I_{3} = \epsilon_{xx}\epsilon_{yy}\epsilon_{zz} + 2\epsilon_{xy}\epsilon_{yz}\epsilon_{zx} - \epsilon_{xx}\epsilon_{yz}^{2} - \epsilon_{yy}\epsilon_{zx}^{2} - \epsilon_{zz}\epsilon_{xy}^{2}$$

$$\frac{\partial \epsilon_k}{\partial \epsilon_{ij}} = \frac{\partial \epsilon_k}{\partial I_1} \frac{\partial I_1}{\partial \epsilon_{ij}} + \frac{\partial \epsilon_k}{\partial I_2} \frac{\partial I_1}{\partial \epsilon_{ij}} + \frac{\partial \epsilon_k}{\partial I_3} \frac{\partial I_1}{\partial \epsilon_{ij}}$$

### Isotropic damage: stress update

**PyFEM** 

```
def getStress( self, kinematics ):
```

```
kappa = self.getHistoryParameter('kappa')
```

```
eps , detadstrain = self.getEquivStrain( kinematics.strain )
```

```
if eps > kappa:
    progDam = True
    kappa = eps
else:
    progDam = False
```

```
self.setHistoryParameter( 'kappa', kappa )
```

```
omega , domegadkappa = self.getDamage( kappa )
```

```
effStress = dot( self.De , kinematics.strain )
```

```
stress = ( 1. - omega ) * effStress
tang = ( 1. - omega ) * self.De
```

```
if progDam:
  tang += -domegadkappa * outer( effStress , detadstrain )
```

```
return stress , tang
```

### One-dimensional elasto-plasticity

def getStress( self, deformation ):

```
# retrieve history variables
                                               two history variables/GP
epsp0 = self.getHistoryParameter('plasticStr')
alpha0 = self.getHistoryParameter('hardening')
sigmaTrial = self.E * ( deformation.strain - epsp0 )
yieldFunc = abs ( sigmaTrial ) - ( self.sigmaY + self.K * alpha0 )
if yieldFunc <= 0.:
                                        isotropic hardening
 sigma = sigmaTrial
 tang = self.E
 epsp = epsp0
 alpha = alpha0
else:
 deltaGamma = yieldFunc / (self.E + self.K)
 sigma = ( 1. - deltaGamma * self.E / abs (sigmaTrial) ) * sigmaTrial
 epsp = epsp0 + deltaGamma * sign ( sigmaTrial )
 alpha = alpha0 + deltaGamma
 tang = self.E * self.K / ( self.E + self.K )
self.setHistoryParameter( 'plasticStr', epsp
self.setHistoryParameter( 'hardening', alpha )
return sigma, tang
```

It has been mathematically proved, full NR converges quadratically closed to the solution.

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In reality ...

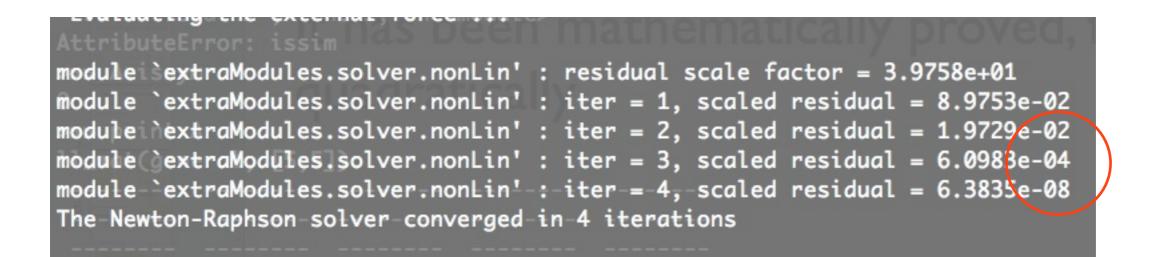
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### In reality ...

AttributeError: issim
<pre>moduleisextraModules.solver.nonLin' : residual scale factor = 3.9758e+01</pre>
<pre>module `extraModules.solver.nonLin' : iter = 1, scaled residual = 8.9753e-02</pre>
<pre>modulei`extraModules.solver.nonLin' : iter = 2, scaled residual = 1.9729e-02</pre>
<pre>module@extraModules.solver.nonLin' : iter = 3, scaled residual = 6.0983e-04</pre>
<pre>module-`extraModules.solver.nonLin'-:-iter-=-4,-scaled residual = 6.3835e-08</pre>
The-Newton-Raphson-solver-converged-in-4 iterations

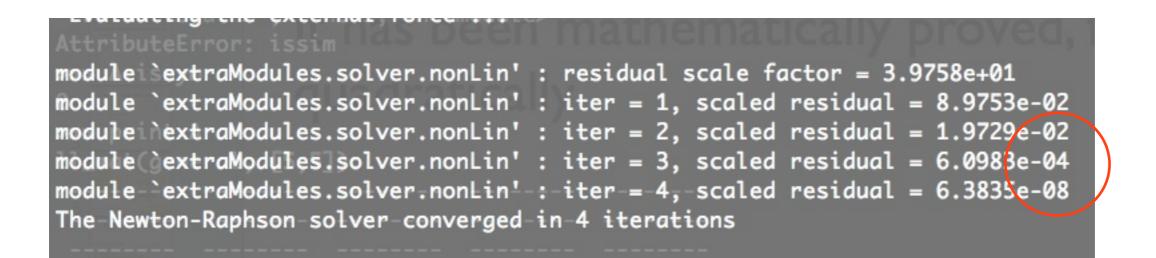
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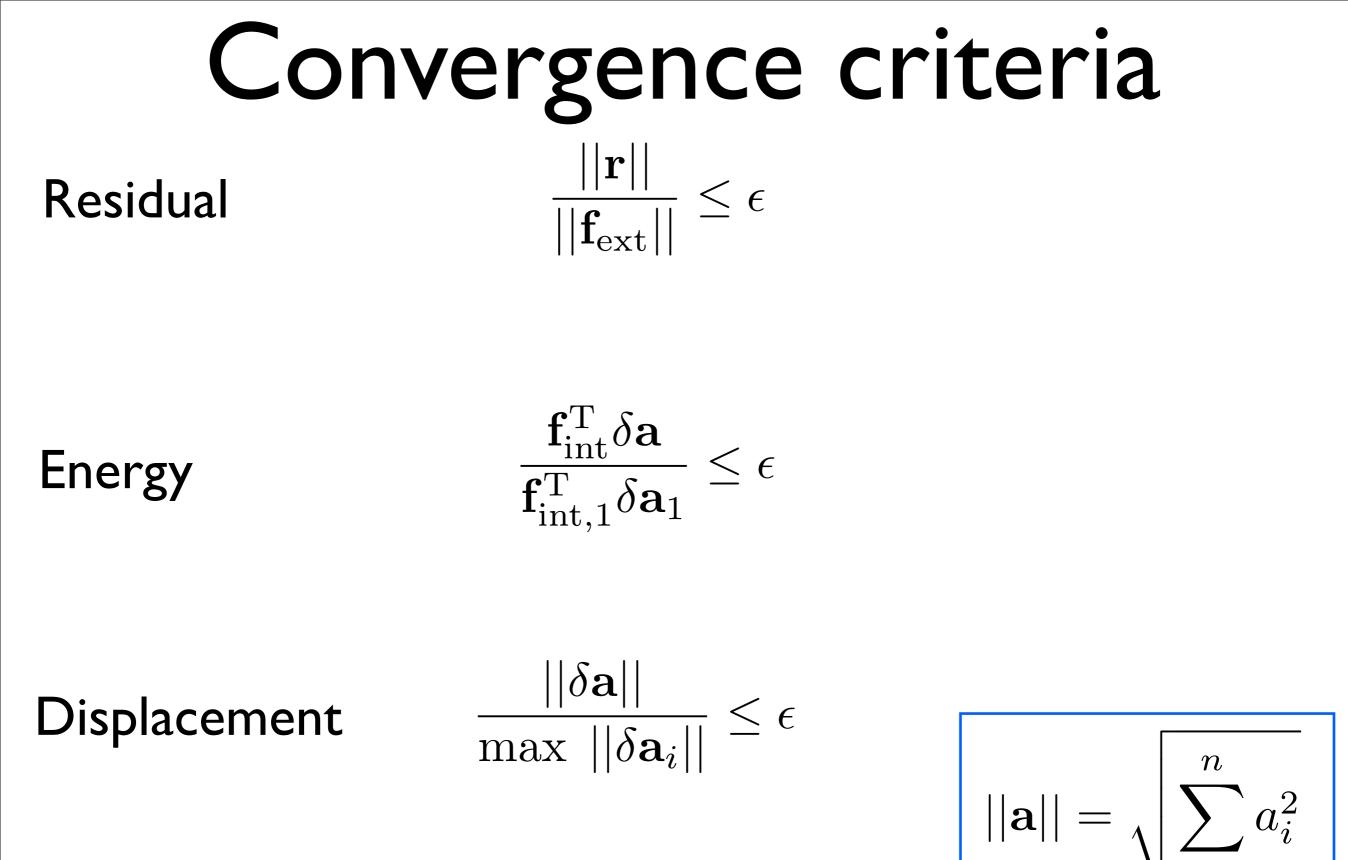


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In reality ...



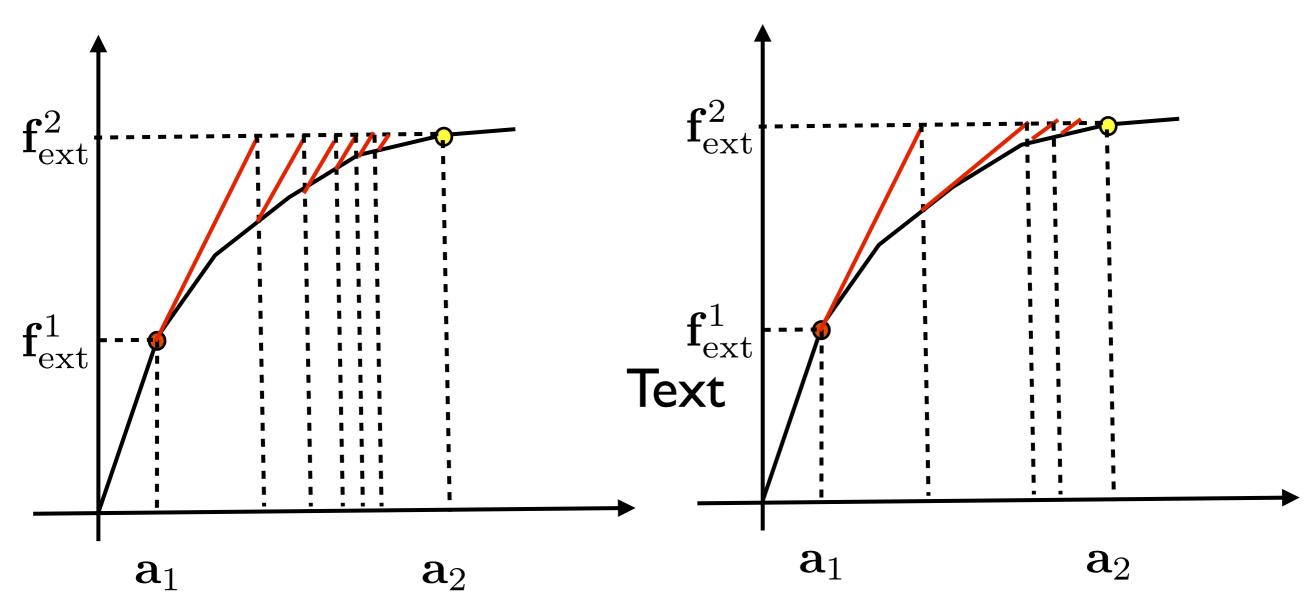
quadratic



#### relative norms

tolerance is problem dependent

### Modified NR methods



Initial stiffness method Modified Newton-Raphson more iterations per step without consistent material tangent

$$\mathbf{f}_{int}(\mathbf{a}) = \mathbf{0}$$
  
 $a_5 = \lambda \bar{a}, \ \lambda = 0, 1, 2, ...$ 

For each load step:

First iteration:  $\mathbf{K}\delta\mathbf{a}_{(1)} = -\mathbf{f}_{\text{int}}$  $\delta a_5 = \bar{a}$ 

From second iteration:

$$\mathbf{K}\delta\mathbf{a}_{(i)} = -\mathbf{f}_{\text{int}}$$
$$\delta a_5 = 0$$

Argyris (1965) Batoz, Dhatt (1979) f<sub>ext</sub> ↓ softening materials a

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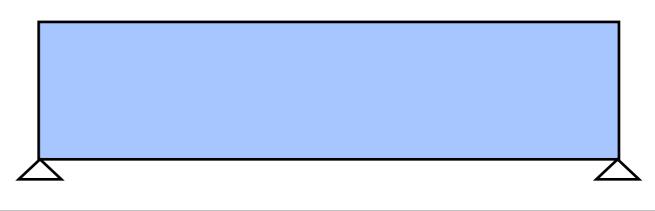
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# Divergence issues

<pre>moduleisextraModules.solver.nonLin' : residual scale factor = 4.2697e+01</pre>
modulea`extraModules.solver.nonLin' : iter = 1, scaled residual = 1.1246e-01
module `extraModules.solver.nonLin!e: iter = 2, scaled residual = 5.5655e-02
moduleutextraModulesusolver.nonLin' : iter = 3, scaled residual = 3.1334e-02
moduleisextraModules.solver.nonLin' : iter = 4, scaled residual = 3.9274e-02
module `extraModules.solver.nonLin' : iter = 5, scaled residual = 5.4948e-02
moduleinextraModules.solver.nonLin' : iter = 6, scaled residual = 3.3929e-02
module @extraModules.solver.nonLin' : iter = 7, scaled residual = 4.6461e-02
module-`extraModules.solver.nonLin'-:-iter-=-8,-scaled residual = 2.7252e-02
module_`extraModules.solver.nonLin'-:-iter-=-9,-scaled residual = 4.7695e-02
module `extraModules.solver.nonLin' :- iter = 10, -scaled residual = 4.1914e-02
module `extraModules.solver.nonLin'-:-iter-=-11,-scaled residual = 1.1236e-01
module - extraModules.solver.nonLin' -:- iter =- 12, - scaled residual = 6.0917e-02
module `extraModules.solver.nonLin' : iter = 13, scaled residual = 5.6249e-02
module `extraModules.solver.nonLin' : iter = 14, scaled residual = 3.4289e-02
module `extraModules.solver.nonLin' : iter = 15, scaled residual = 4.4957e-02
module )extraModules.solver.nonLin' ; iter = 16, scaled residual = 2.7413e-02

### the system is failed structurally or numerically???

# Elasto-plastic models

## Linear visoelasticity