

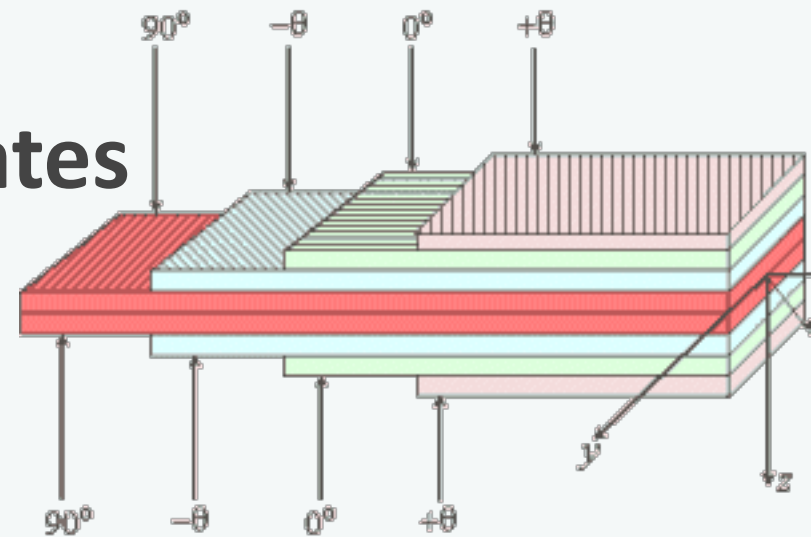
Isogeometric cohesive interface elements for 2D/3D delamination analysis

Vinh Phu NGUYEN, Pierre KERFRIDEN, Stéphane P.A. BORDAS

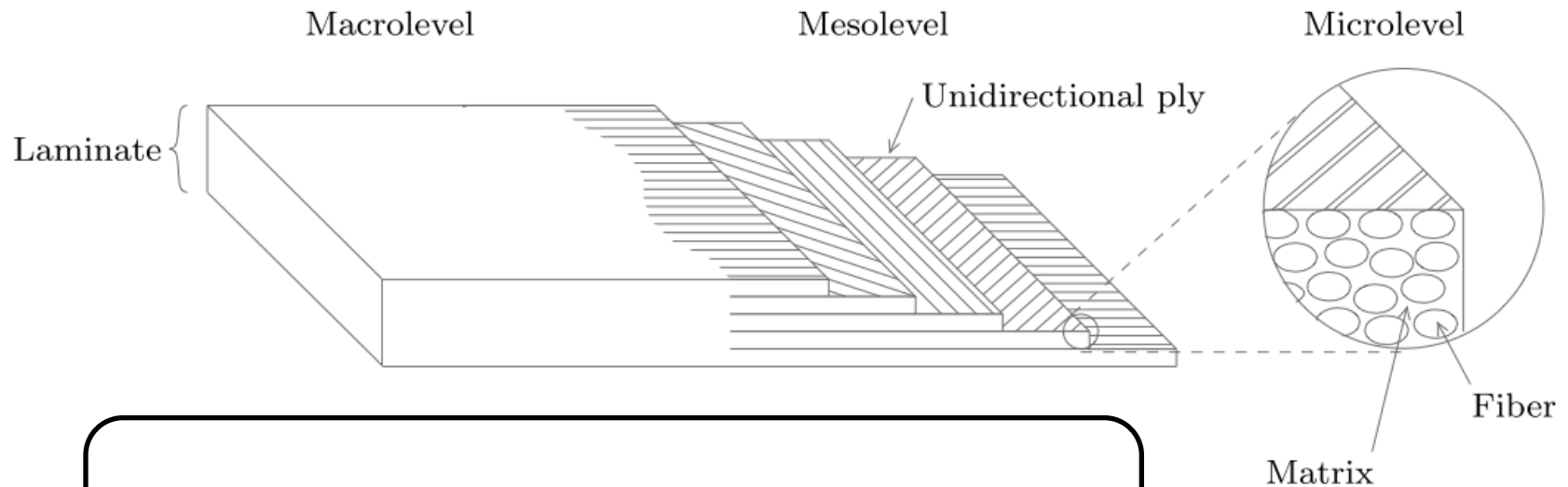
Institute of Mechanics and Advanced Materials

Cardiff University, Wales, UK

Failure of composite laminates



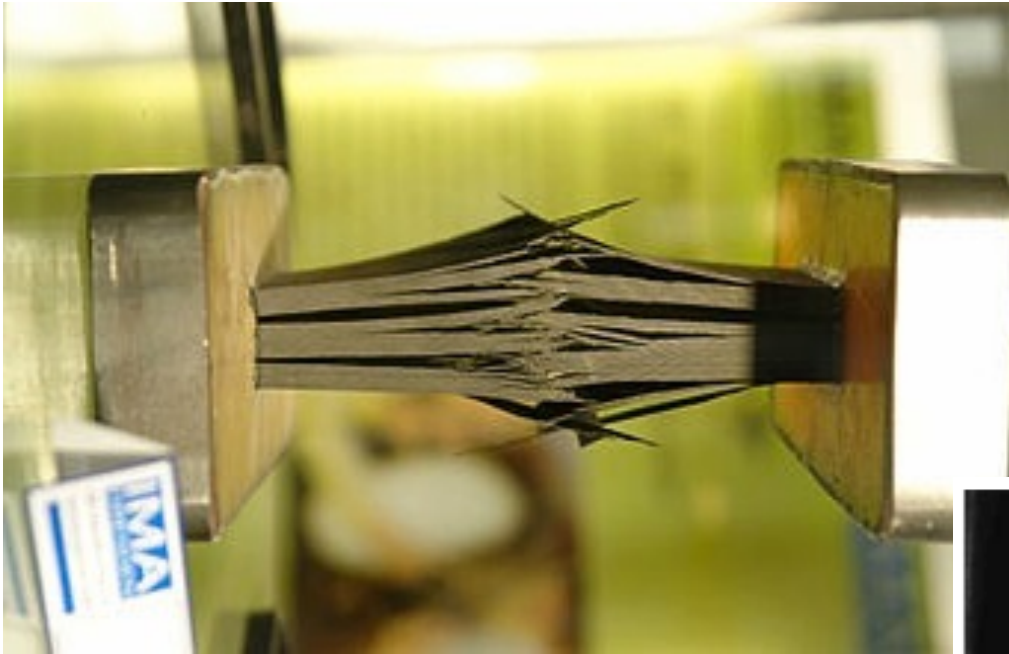
Three levels of observation for composite laminates



- Mesolevel
- Unidirectional ply: orthotropic materials

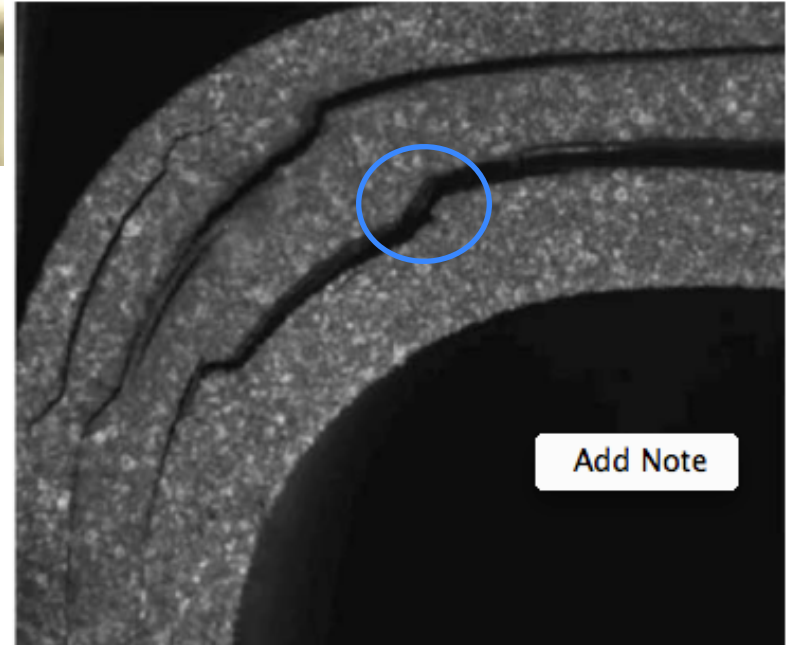
FP van der Meer, Mesolevel Modeling of Failure in Composite Laminates: Constitutive, Kinematic and Algorithmic Aspects, Arch Comput Methods Eng (2012) 19:381–425.

Failure modes of composite laminates



delamination
(interlaminar cracking)

matrix failure
(intralaminar cracking)



Finite Element Method (FEM)

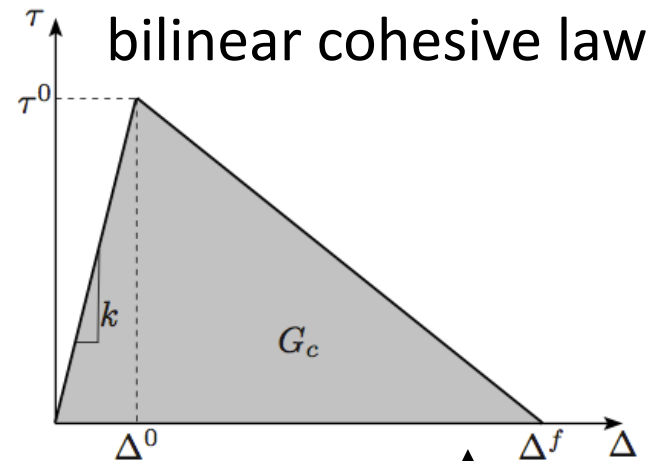
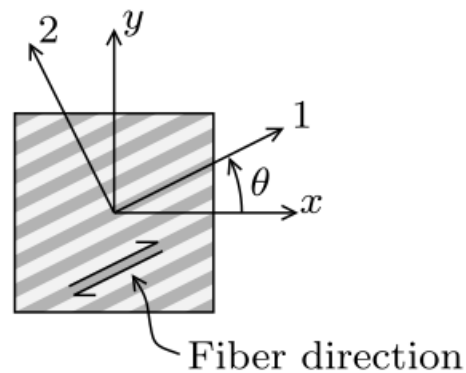
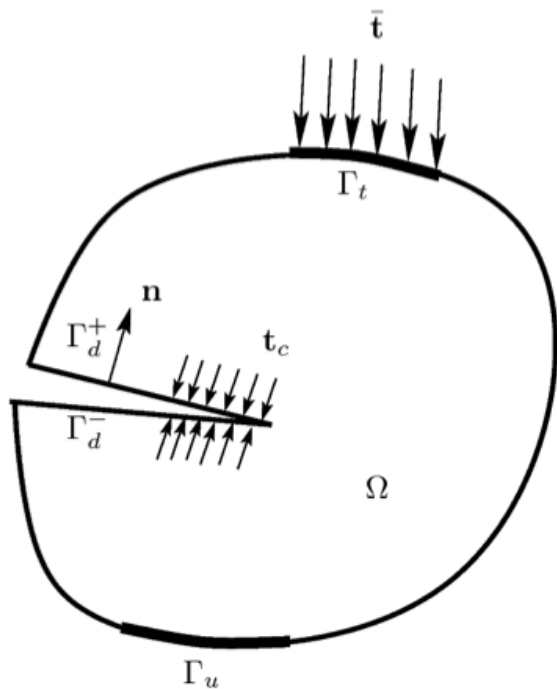
VCCT (Virtual Crack Closure Technique)

- Linear Elastic Fracture Mechanics
- Only for delamination growth
- Not computationally intensive

Cohesive interface elements

- Cohesive zone models
- Delamination initiation/growth
- Computationally intensive/robustness issue
- Decohesion elements, cohesive zones, cohesive elements...

Cohesive cracks weak form



$$\bar{\boldsymbol{\sigma}} = \bar{\mathbf{D}} \bar{\boldsymbol{\epsilon}}$$

Unknown field is the displacement \mathbf{u}

$$\int_{\Omega} \delta \mathbf{u} \cdot \mathbf{b} d\Omega + \int_{\Gamma_t} \delta \mathbf{u} \cdot \bar{\mathbf{t}} d\Gamma_t = \int_{\Omega} \delta \boldsymbol{\epsilon} : \boldsymbol{\sigma}(\mathbf{u}) d\Omega + \int_{\Gamma_d} \delta [[\mathbf{u}]] \cdot \mathbf{t}^c([[\mathbf{u}]]) d\Gamma_d$$

Interface elements

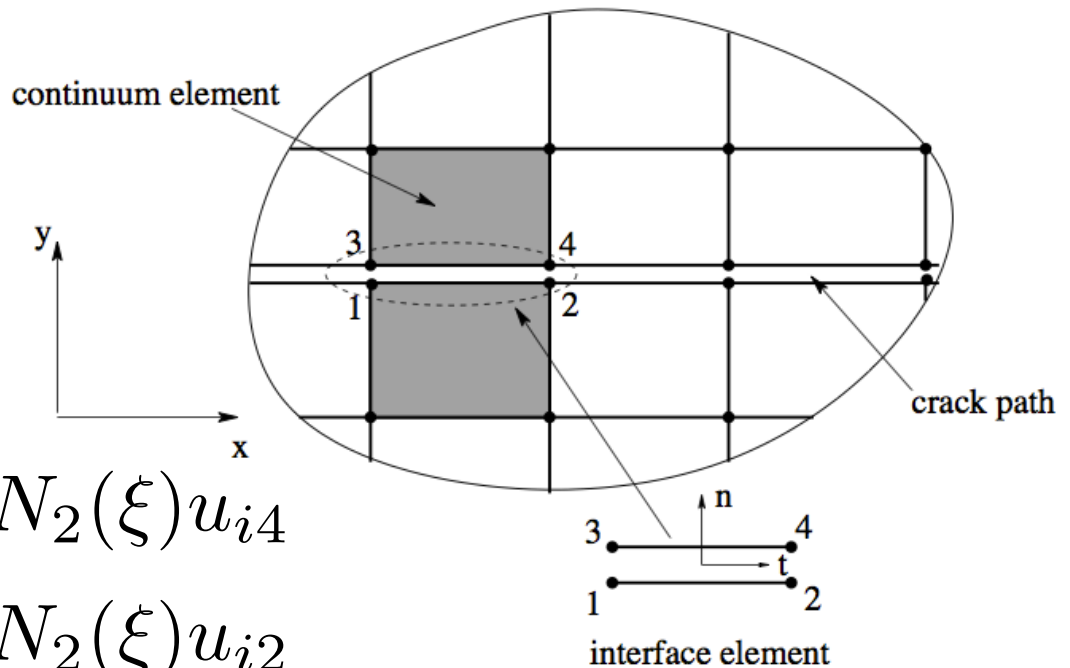
$$\int_{\Omega} \delta \mathbf{u} \cdot \mathbf{b} d\Omega + \int_{\Gamma_t} \delta \mathbf{u} \cdot \bar{\mathbf{t}} d\Gamma_t = \int_{\Omega} \delta \boldsymbol{\epsilon} : \boldsymbol{\sigma}(\mathbf{u}) d\Omega + \int_{\Gamma_d} \delta [[\mathbf{u}]] \cdot \mathbf{t}^c([[\mathbf{u}]]) d\Gamma_d$$

Delamination is a problem in which crack path is known in advance.

$$[[\mathbf{u}]]_i = u_i^+ - u_i^-$$

$$u_i^+ = N_1(\xi)u_{i3} + N_2(\xi)u_{i4}$$

$$u_i^- = N_1(\xi)u_{i1} + N_2(\xi)u_{i2}$$

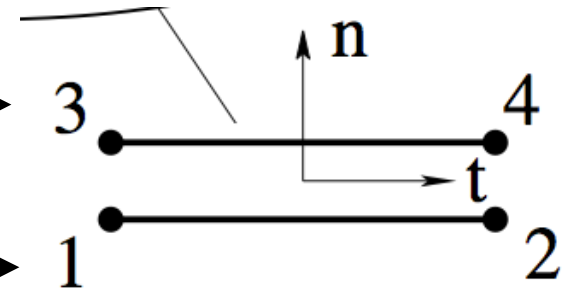


Interface elements: internal force vectors

$$\mathbf{f}^{\text{ext}} = \mathbf{f}^{\text{int}} + \mathbf{f}^{\text{coh}}$$

$$\mathbf{f}^{\text{int}} = \int_{\Omega} \mathbf{B}^T \boldsymbol{\sigma} d\Omega \quad \mathbf{N}^{\text{int}} = \begin{bmatrix} N_1 & 0 & N_2 & 0 \\ 0 & N_1 & 0 & N_2 \end{bmatrix}$$

$$\mathbf{f}^{\text{ext}} = \int_{\Gamma_t} \mathbf{N}^T \bar{\mathbf{t}} d\Gamma$$

$$\mathbf{f}_{ie,+}^{\text{coh}} = \int_{\Gamma} \mathbf{N}_{\text{int}}^T \mathbf{t}^c d\Gamma$$


The diagram shows a rectangular interface element with four nodes labeled 1, 2, 3, and 4. Nodes 1 and 2 are at the bottom, and nodes 3 and 4 are at the top. A normal vector \mathbf{n} points upwards from the center of the element, and a tangent vector \mathbf{t} points to the right. A line with a slash indicates a cut or interface through the element.

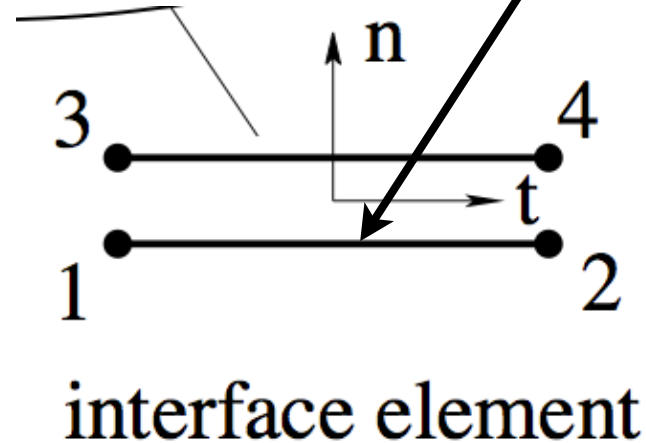
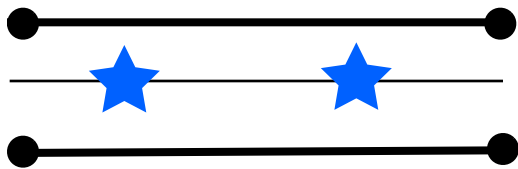
$$\mathbf{f}_{ie,-}^{\text{coh}} = - \int_{\Gamma} \mathbf{N}_{\text{int}}^T \mathbf{t}^c d\Gamma$$

interface element

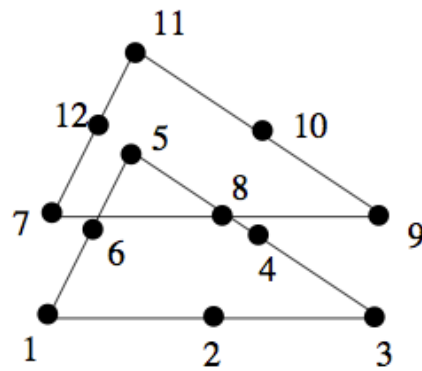
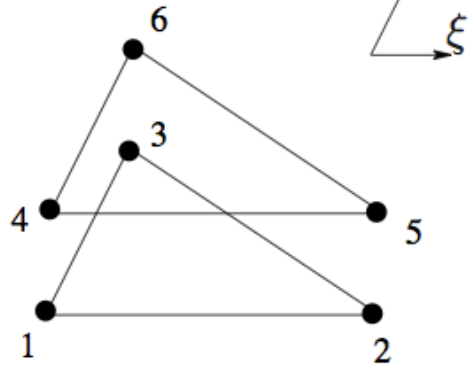
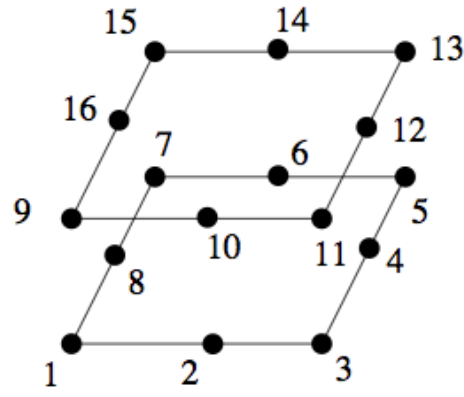
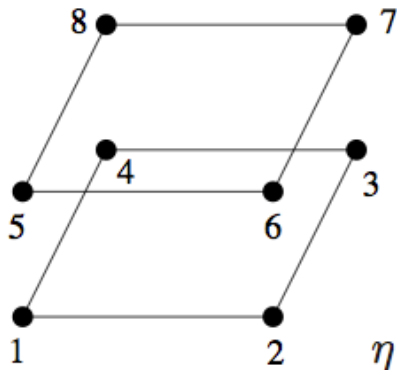
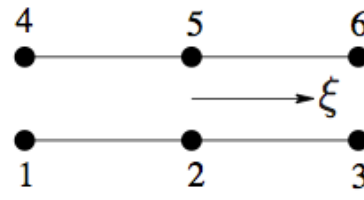
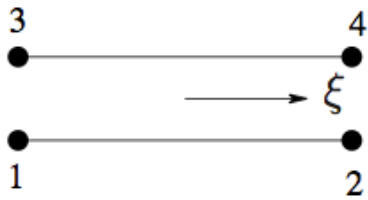
Interface elements: tangent matrix

$$\mathbf{K}_e^{int} = \begin{bmatrix} \int_{\Gamma} \mathbf{N}^T \mathbf{Q}^T \mathbf{T} \mathbf{Q} \mathbf{N} d\Gamma & - \int_{\Gamma} \mathbf{N}^T \mathbf{Q}^T \mathbf{T} \mathbf{Q} \mathbf{N} d\Gamma \\ - \int_{\Gamma} \mathbf{N}^T \mathbf{Q}^T \mathbf{T} \mathbf{Q} \mathbf{N} d\Gamma & \int_{\Gamma} \mathbf{N}^T \mathbf{Q}^T \mathbf{T} \mathbf{Q} \mathbf{N} d\Gamma \end{bmatrix}$$

Numerical integration



Common interface elements

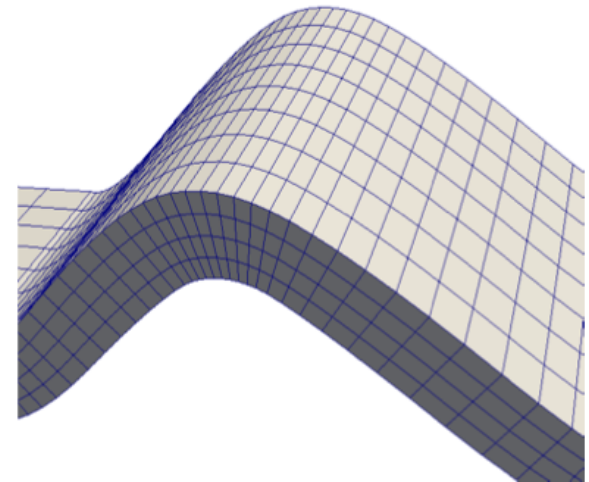


2D

3D

What is wrong with standard interface elements?

- time consuming pre-processing (generation of interface el.)
- no link to CAD data: not ideal for design-analysis cycles
- standard low order Lagrange elements: poor derivative fields such as stresses => very fine mesh in front of the crack tip
- geometry: not exactly represented



Isogeometric interface elements

- fast pre-processing: automatic generation of interface el.
- link to CAD data: ideal for design-analysis cycles
- high order NURBS elements: highly accurate derivative fields
- less expensive than low order Lagrange elements
- geometry: exactly represented

There are no free lunch. However let talk about the good news first.

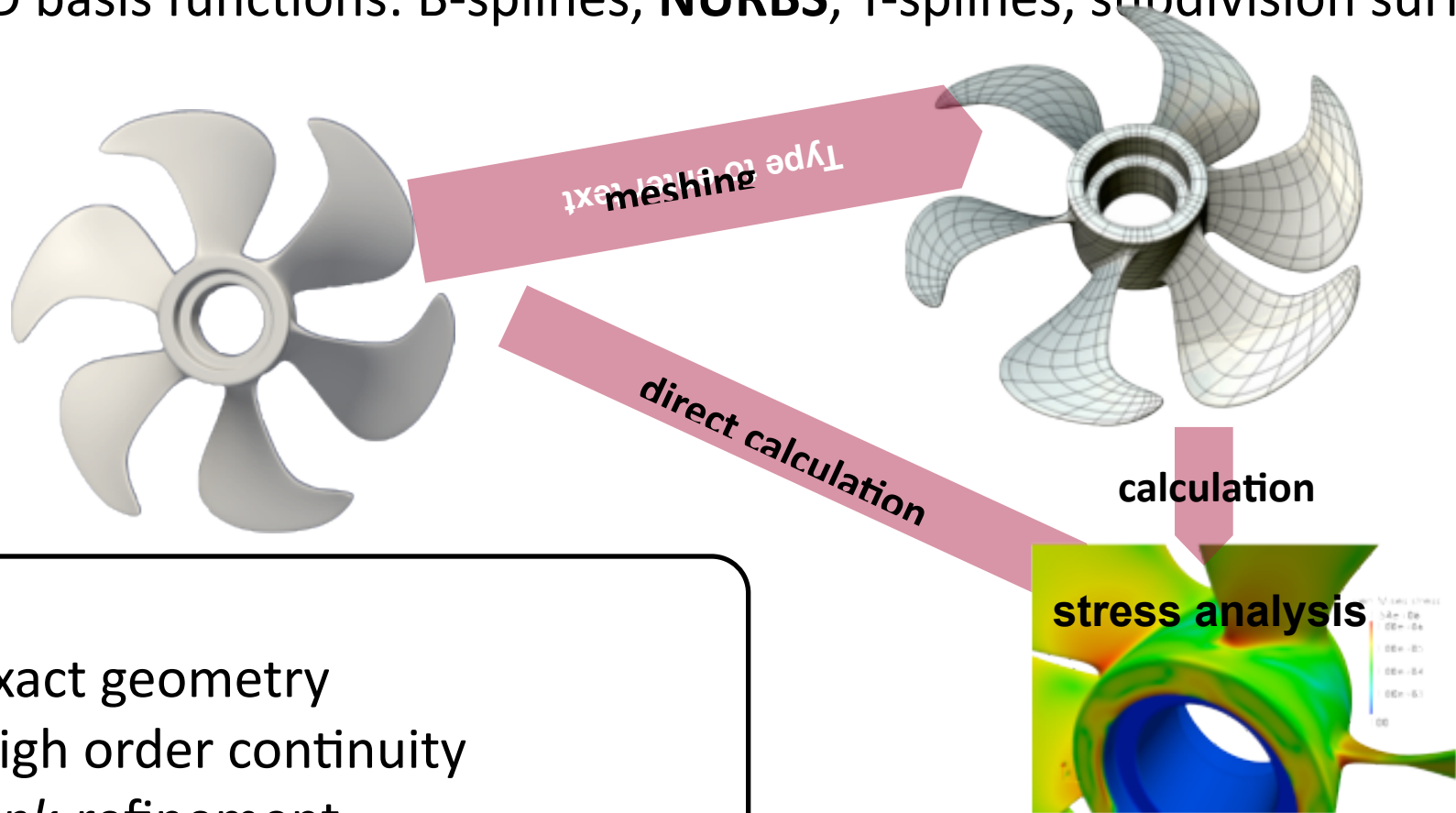
Isogeometric analysis

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Isogeometric analysis

Approximate the unknown fields with the basis functions used to generate the CAD model.

CAD basis functions: B-splines, **NURBS**, T-splines, subdivision surfaces...



- Exact geometry
- High order continuity
- *hpk*-refinement

- P. Kagan, A. Fischer, and P. Z. Bar-Yoseph. New B-Spline Finite Element approach for geometrical design and mechanical analysis. *IJNME*, 41(3):435–458, 1998.
- F. Cirak, M. Ortiz, and P. Schroder. Subdivision surfaces: a new paradigm for thin-shell finite-element analysis. *IJNME*, 47(12): 2039–2072, 2000.
- Constructive solid analysis: a hierarchical, geometry-based meshless analysis procedure for integrated design and analysis. D. Natekar, S. Zhang, and G. Subbarayan. *CAD*, 36(5): 473--486, 2004.
- T.J.R. Hughes, J.A. Cottrell, and Y. Bazilevs. Isogeometric analysis: CAD, finite elements, NURBS, exact geometry and mesh refinement. *CMAME*, 194(39-41):4135–4195, 2005.
- J. A. Cottrell, T. J.R. Hughes, and Y. Bazilevs. *Isogeometric Analysis: Toward Integration of CAD and FEA*. Wiley, 2009.

B-splines basis functions

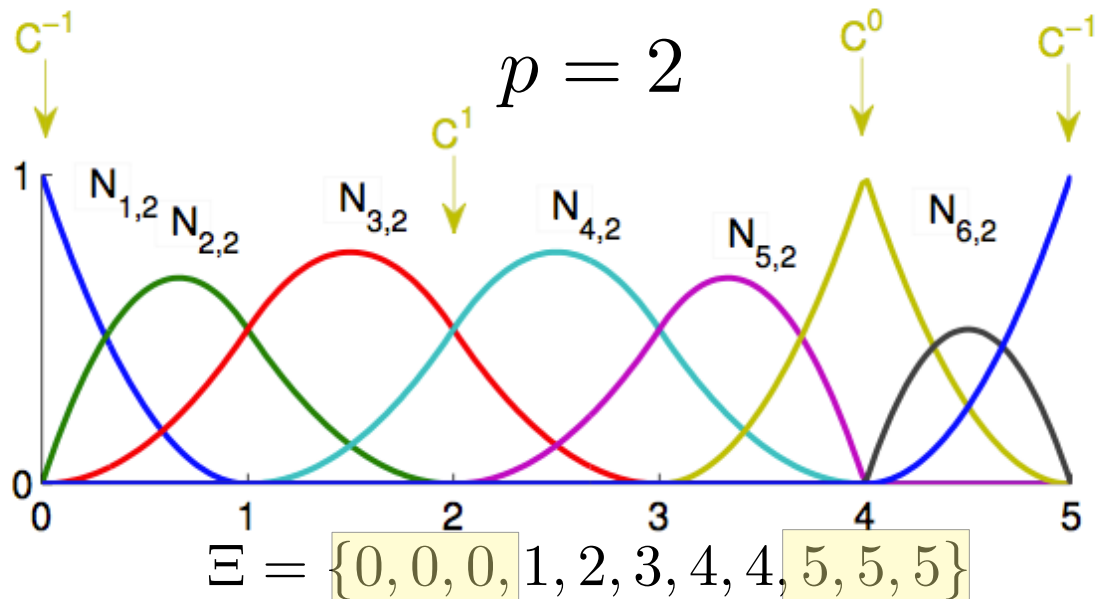
$$\Xi = \{\xi_1, \xi_2, \dots, \xi_{n+p+1}\} \quad \text{knot vector}$$

$$N_{i,0}(\xi) = \begin{cases} 1 & \text{if } \xi_i \leq \xi < \xi_{i+1} \\ 0 & \text{otherwise} \end{cases} \quad \text{\sigma}$$

$$N_{i,p}(\xi) = \frac{\xi - \xi_i}{\xi_{i+p} - \xi_i} N_{i,p-1}(\xi) + \frac{\xi_{i+p+1} - \xi}{\xi_{i+p+1} - \xi_{i+1}} N_{i+1,p-1}(\xi)$$

Properties

- Partition of Unity
- Linear independence
- Non-negativity
- C^{p-m} continuity
- Not interpolants



B-splines

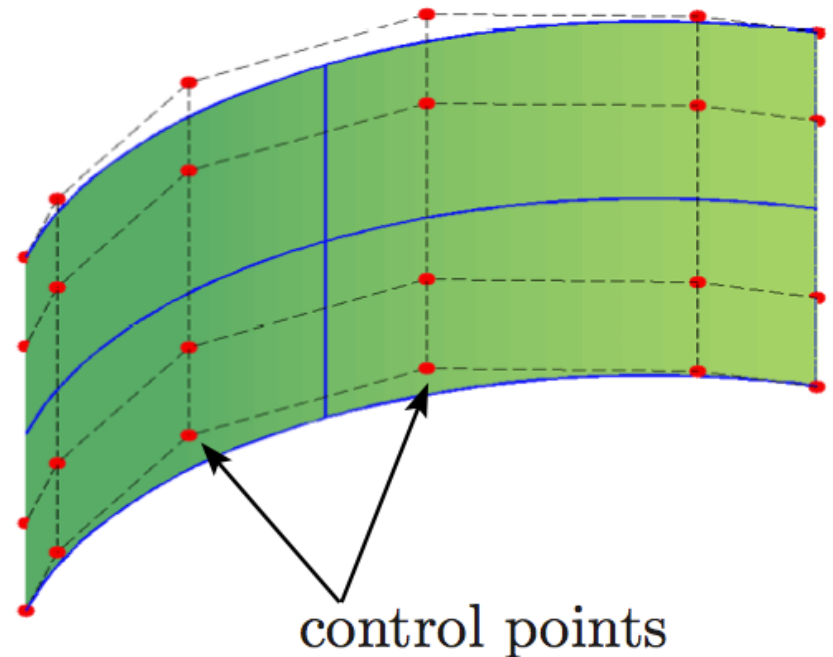
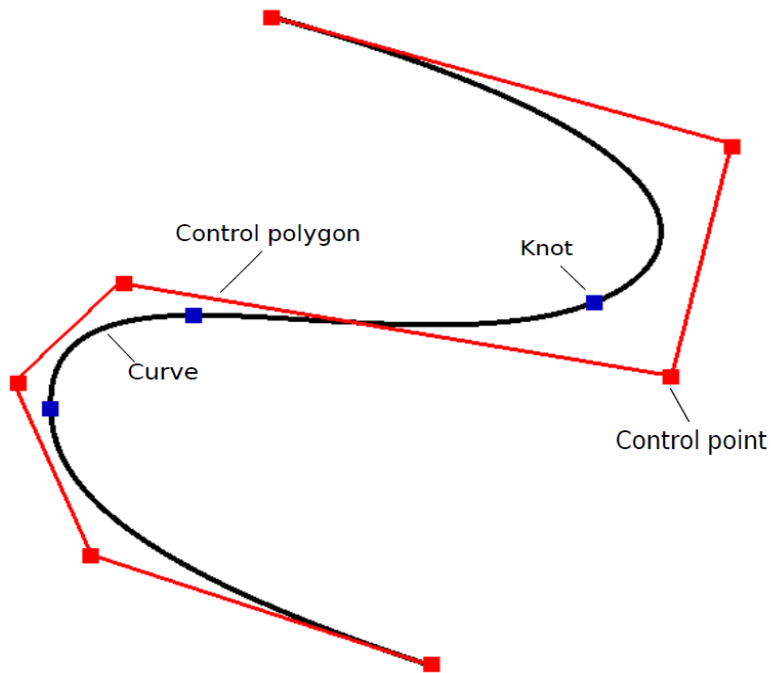
$$\Xi = \{\xi_1, \xi_2, \dots, \xi_{n+p+1}\}$$

$$\Xi^1 = \{\xi_1, \xi_2, \dots, \xi_{n+p+1}\}$$

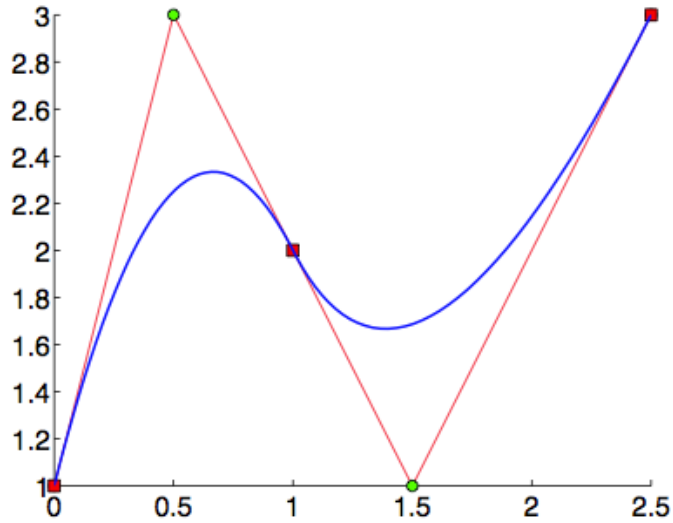
$$\Xi^2 = \{\eta_1, \eta_2, \dots, \eta_{m+q+1}\}$$

$$\mathbf{C}(\xi) = \sum_{i=1}^n N_{i,p}(\xi) \mathbf{B}_i$$

$$\mathbf{S}(\xi, \eta) = \sum_{i=1}^n \sum_{j=1}^m N_{i,p}(\xi) M_{j,q}(\eta) \mathbf{B}_{ij}$$



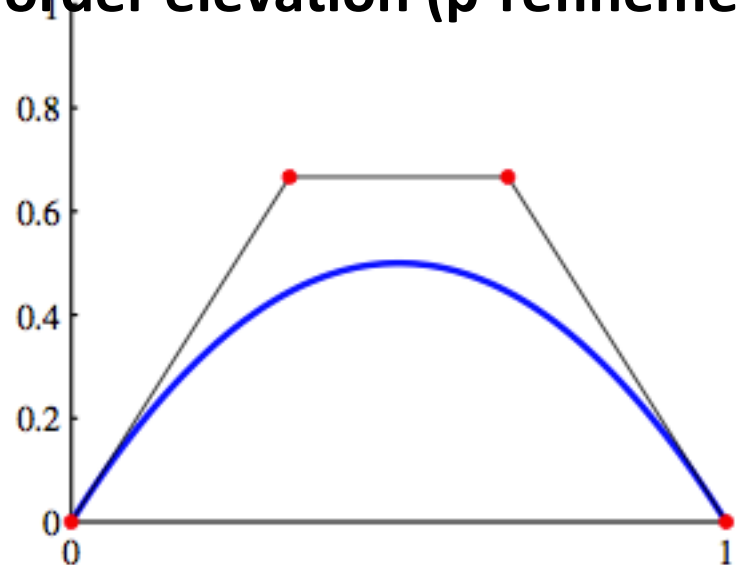
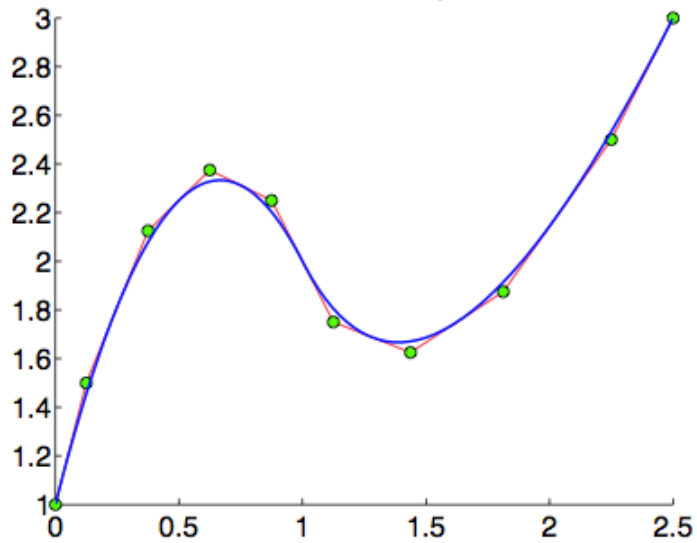
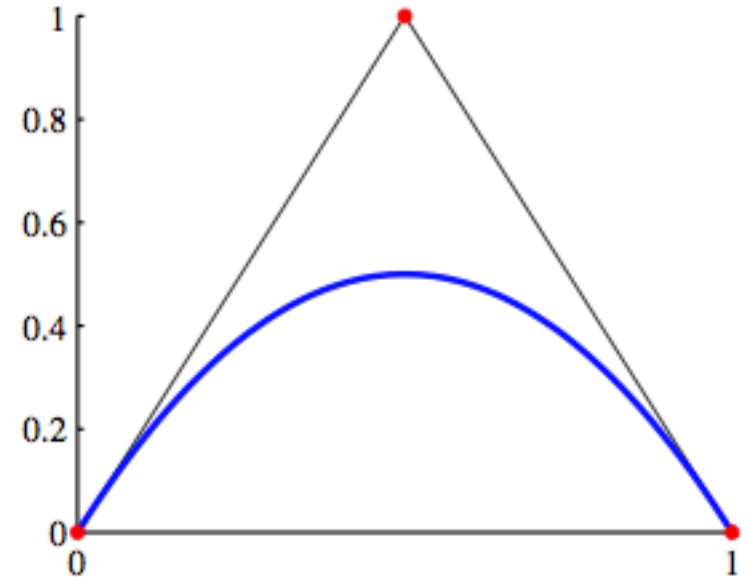
Enriching B-splines



knot insertion (h-refinement)

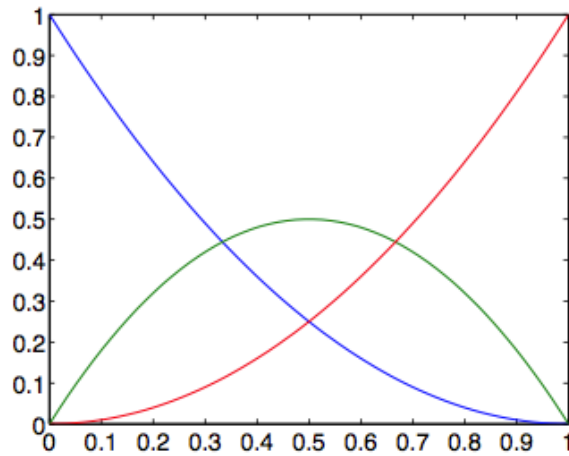
+

order elevation (p-refinement)

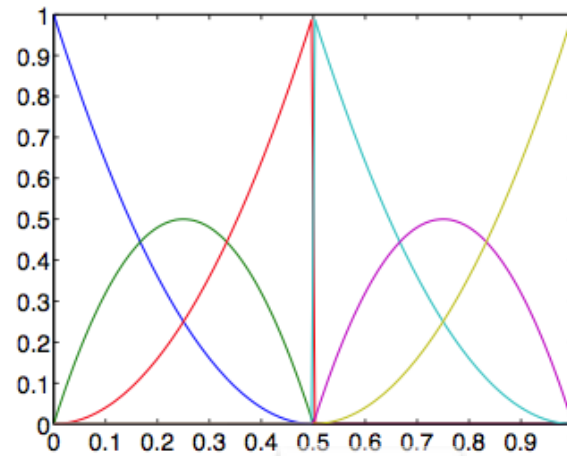


does not change B-splines geometrically/parametrically

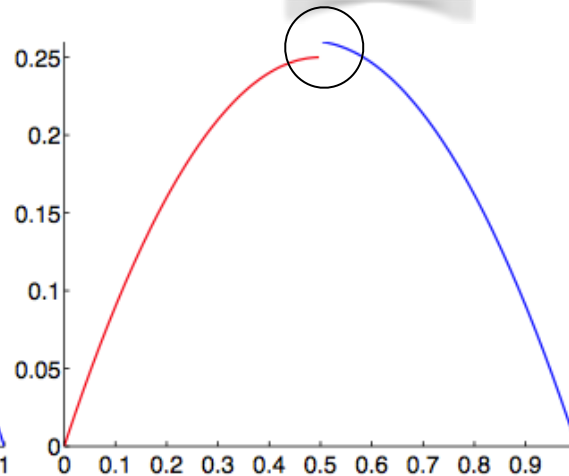
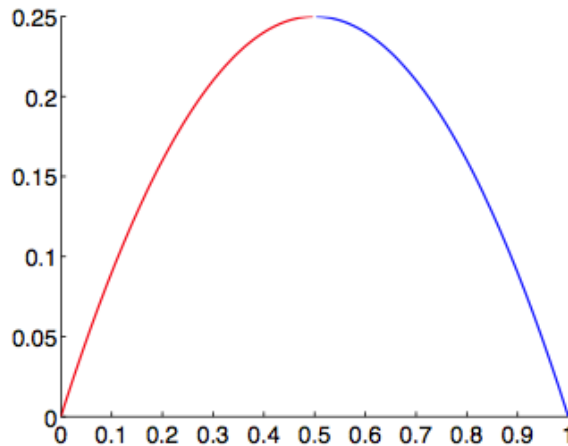
Knot insertion to create discontinuities



(a) $\Xi = \{0, 0, 0, 1, 1, 1\}$



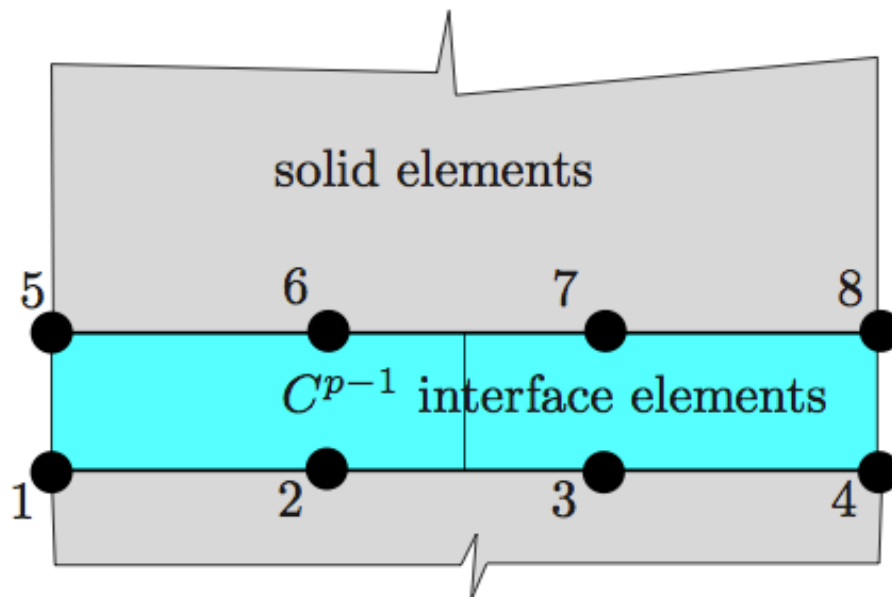
(b) $\Xi' = \{0, 0, 0, 0.5, 0.5, 0.5, 1, 1, 1\}$



$$p = 2$$

does not change B-splines geometrically/parametrically
knot insertion: stable algorithms, available implementations

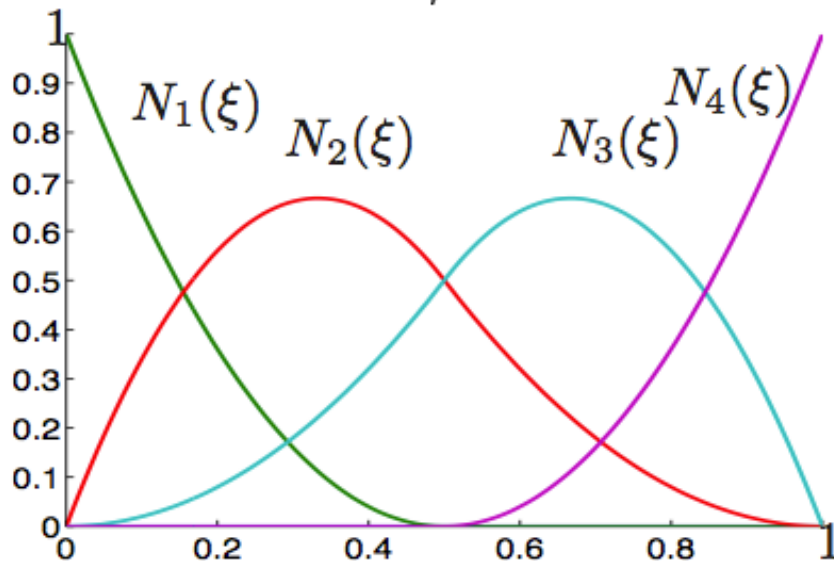
Isogeometric cohesive elements



2 quadratic int. elements

[1,2,3,5,6,7]

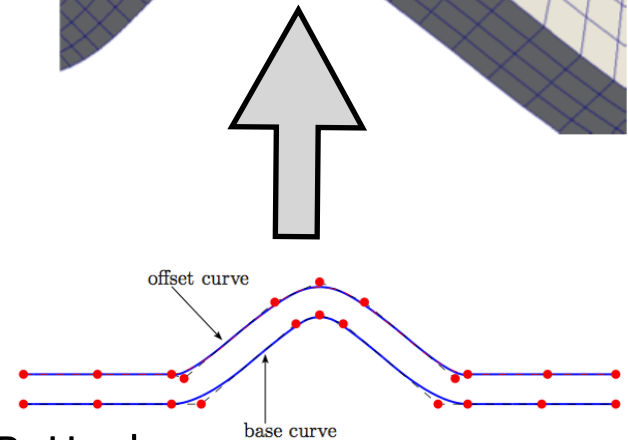
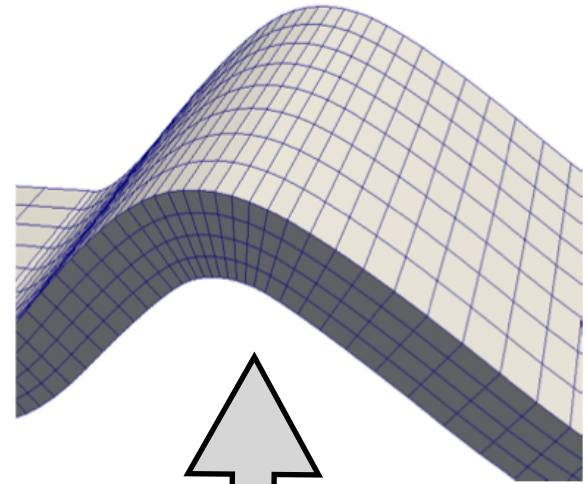
[2,3,4,6,7,8]



automatically generated
using knot insertion.

Isogeometric cohesive elements: advantages

- 2D Mixed mode bending test (MMB)
- 2 x 70 quartic-linear B-spline elements
- run time on a laptop 4GB of RAM: 6 s
- energy arc-length control



1. C. V. Verhoosel, M. A. Scott, R. de Borst, and T. J. R. Hughes. An isogeometric approach to cohesive zone modeling. *IJNME*, 87:336–360, 2011.
2. V. P. Nguyen and H. Nguyen-Xuan. High-order B-splines based finite elements for delamination analysis of laminated composites. *Com. Str.*, 102:261–275, 2013.
3. V.P. Nguyen, P. Kerfriden, S. Bordas. Isogeometric cohesive elements for two and three dimensional composite delamination analysis, 2013, Arxiv.

Examples

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MIGFEM



- quick prototyping
- tutorial codes

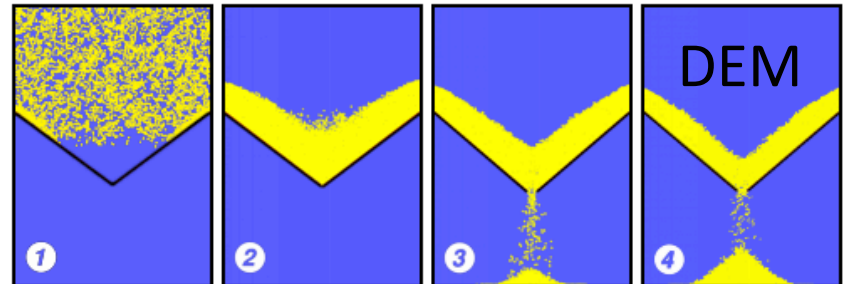
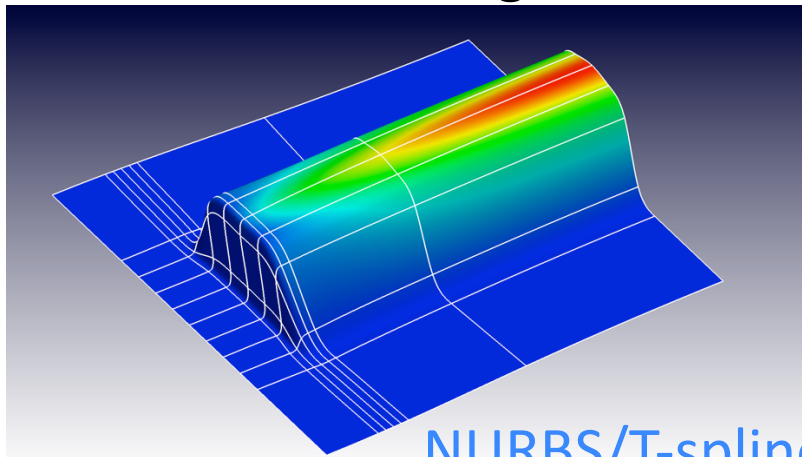
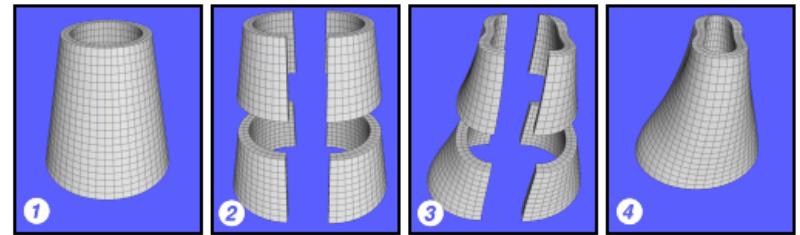
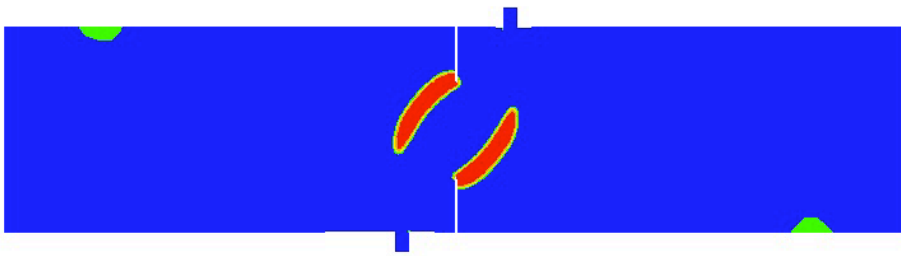
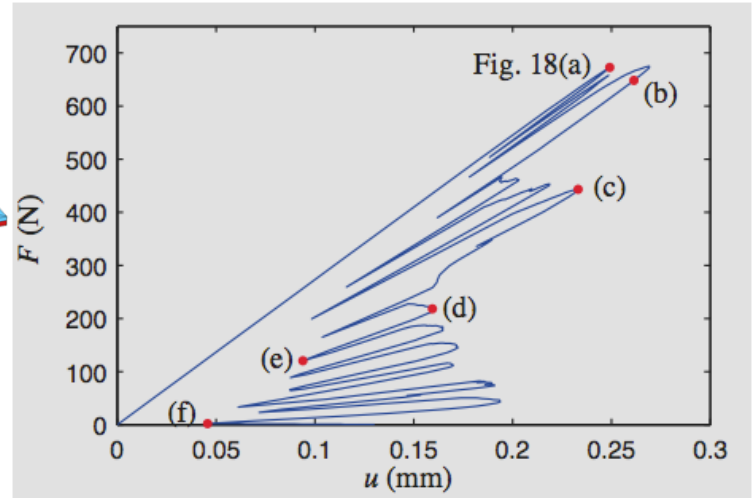
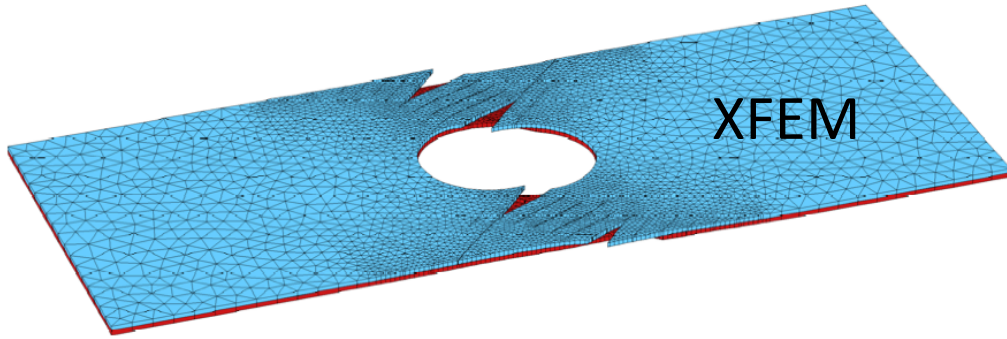
- open source Matlab Isogeometric (X)FEM
- 2D/3D solid mechanics with geometry nonlinearities
- 2D XIGA for LEFM and material interfaces
- Structural mechanics: beam, plate, shells (large deformation)
- <http://sourceforge.net/projects/cmcodes/>

jem-jive (Linux, Mac OS, Windows)

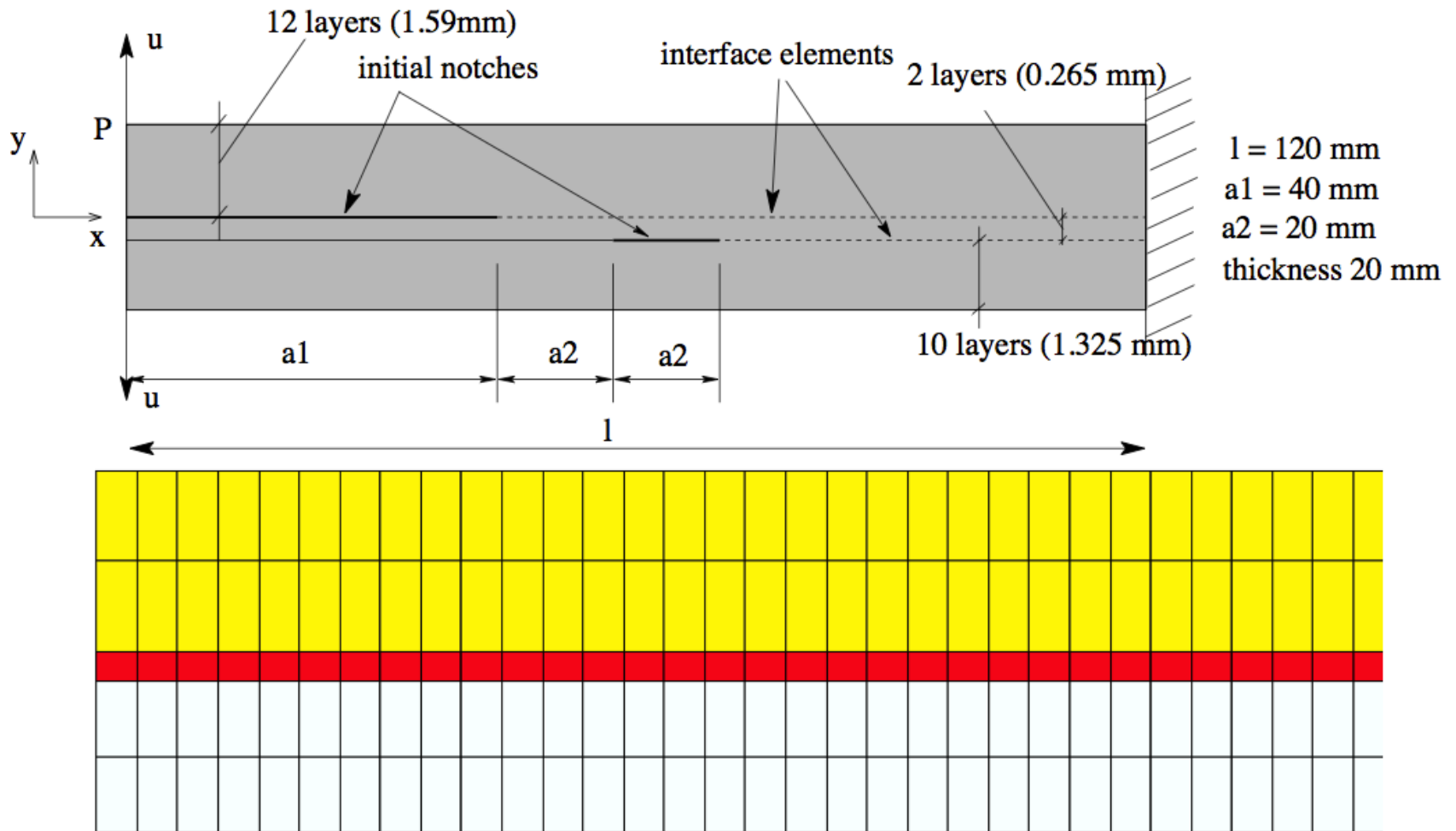
- commercial C++ toolkit for PDEs
- not a general purpose FE package
- tailor made applications, suitable for researchers
- apps: XFEM, dG, IGA, DEM, FVM etc.
- support parallel computing
- implements useful concepts available in other programming languages--Java, Fortran 90, Matlab and C#
- tensor class: useful to evaluating complex constitutive models
- http://www.dynaflow.com/en_GB/jive.html



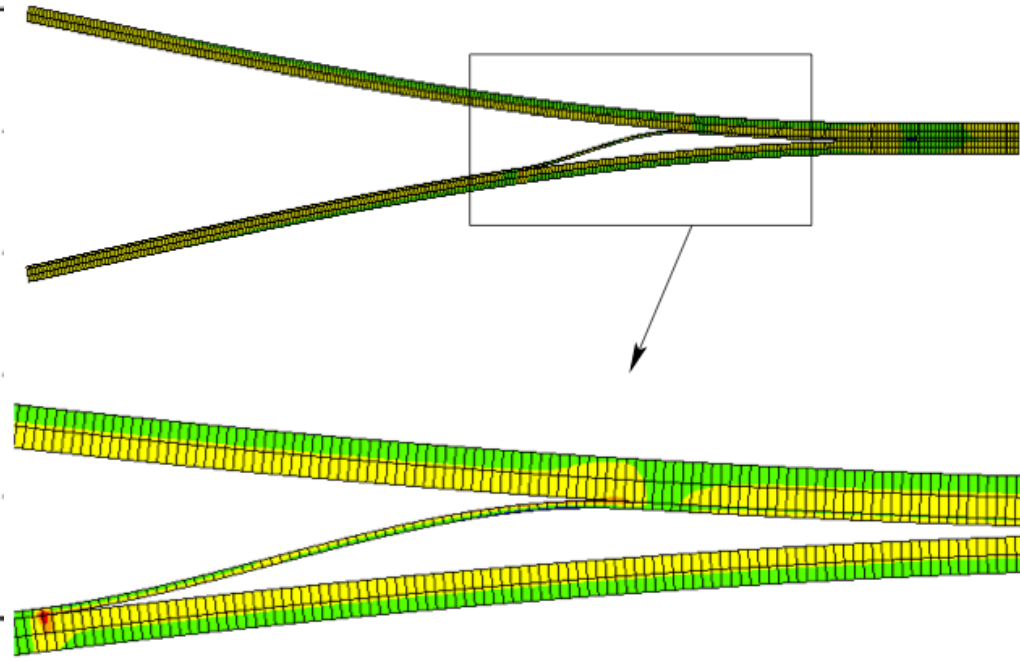
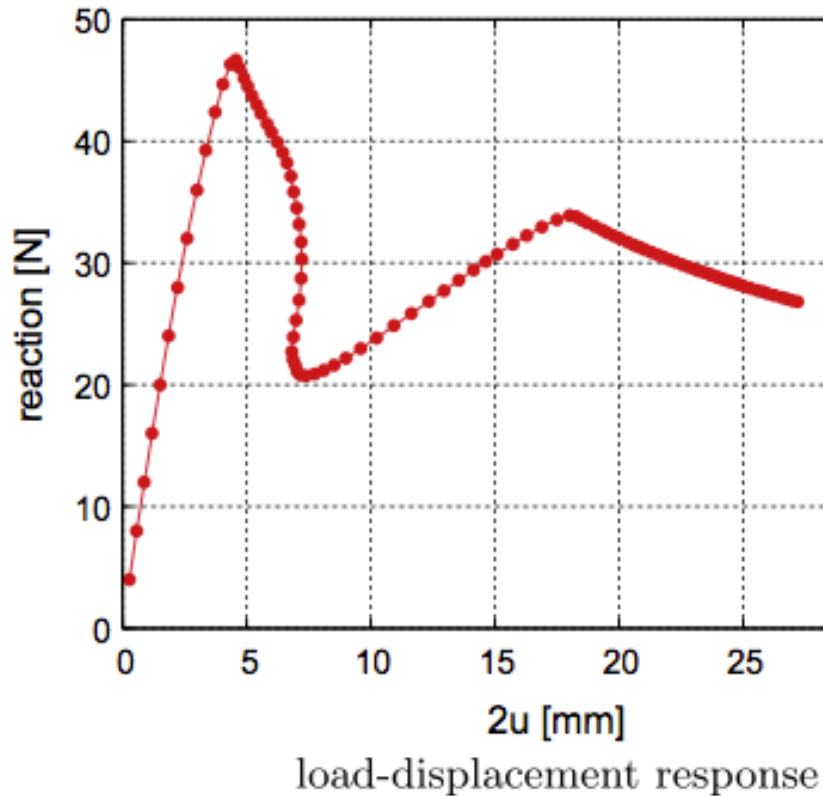
jem-jive: some typical examples



Isogeometric cohesive elements: 2D example

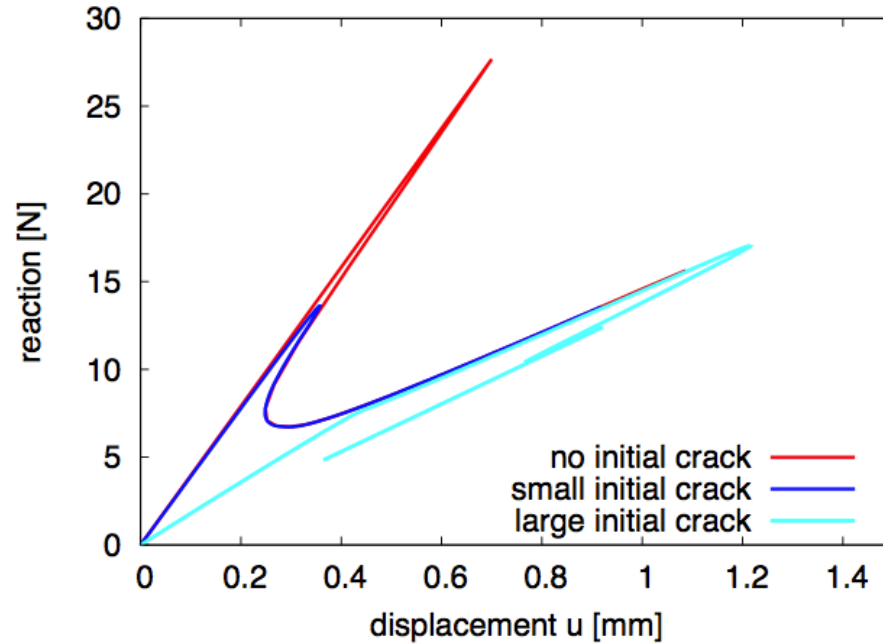
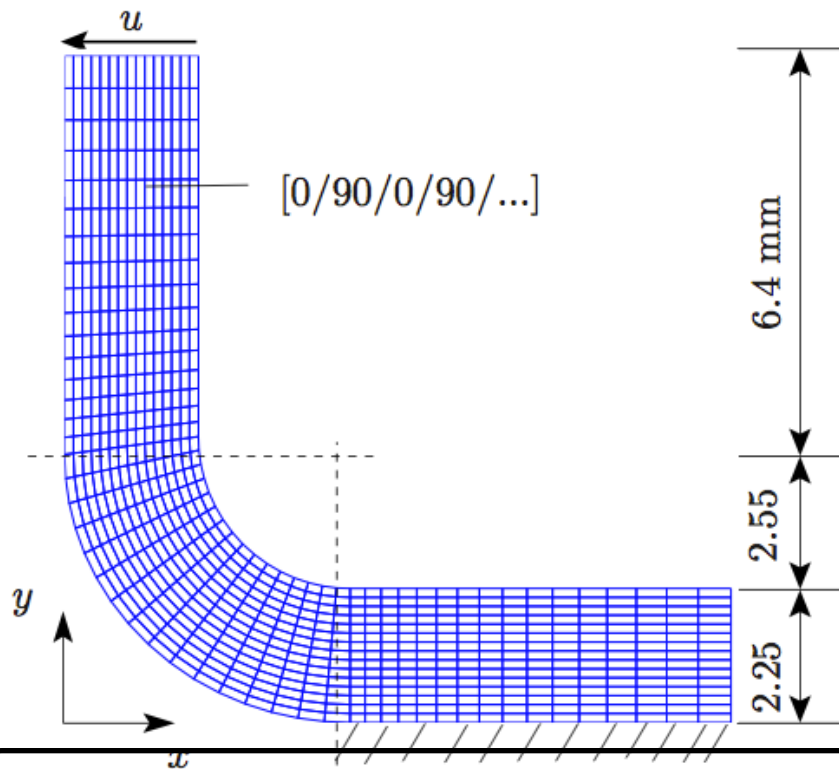


Isogeometric cohesive elements: 2D example



Alfano G, Crisfield MA. Finite element interface models for the delamination analysis of laminated composites: mechanical and computational issues. *IJNME* 2001;50(7):1701–36.

Isogeometric cohesive elements: 2D example

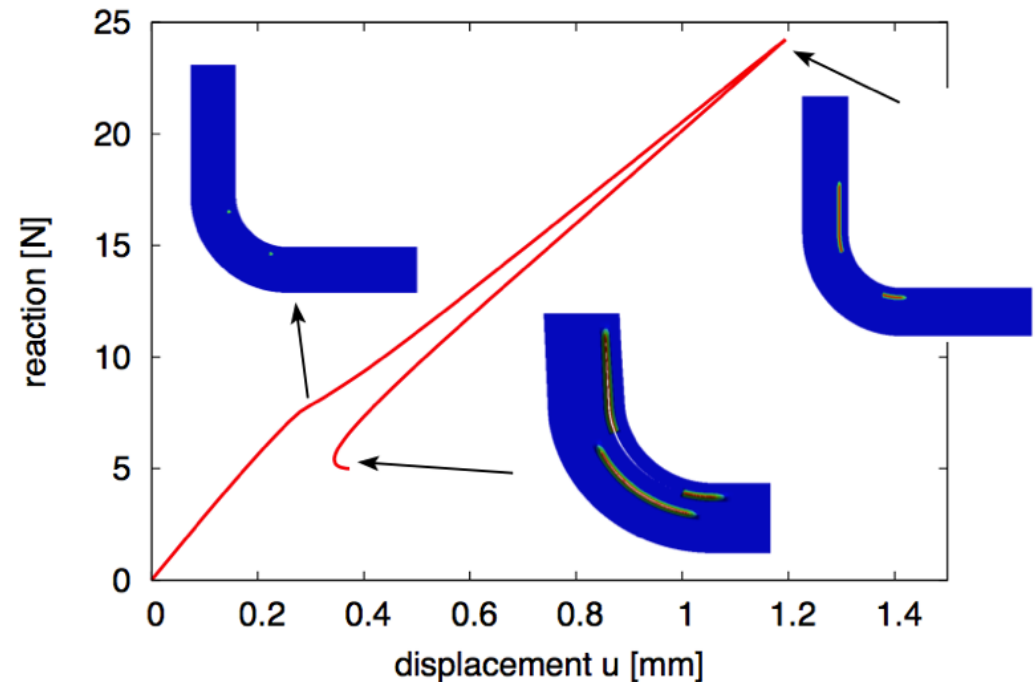


- exact geometry by NURBS
- It is straightforward to vary
 - (1) number of plies and
 - (2) # of interface elements:
- Suitable for parameter studies/design
- Cohesive law: bilinear law of Turon et al. 2006

Isogeometric cohesive elements: 2D example

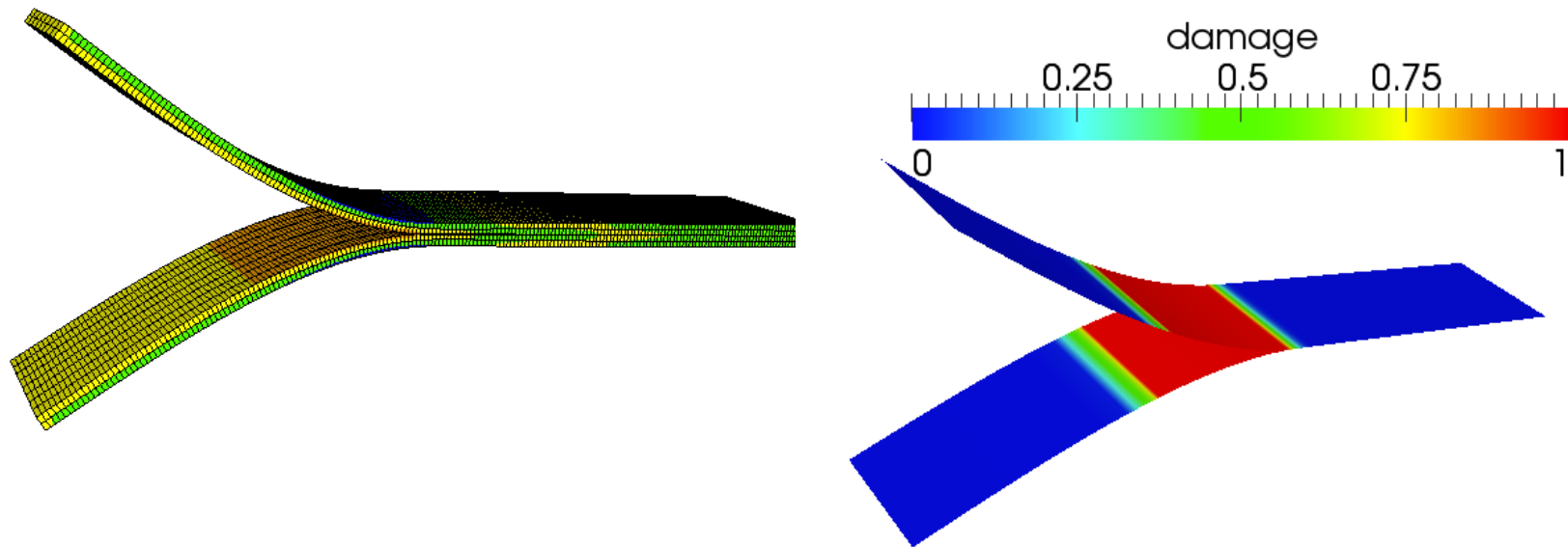


M. A. Gutierrez. Energy release control for numerical simulations of failure in quasi-brittle solids. Communications in Numerical Methods in Engineering, 20(1):19–29, 2004



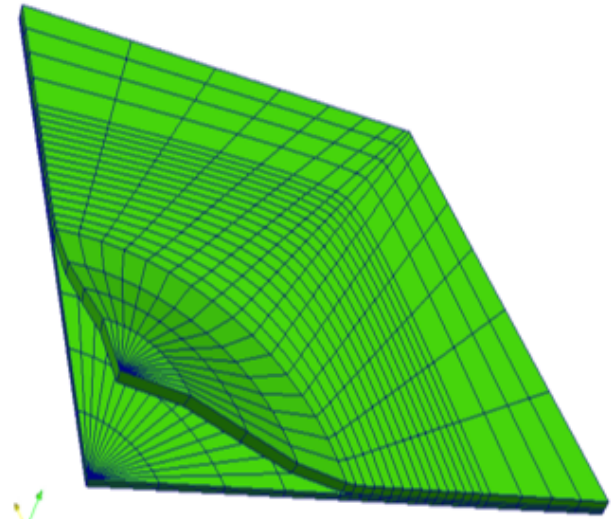
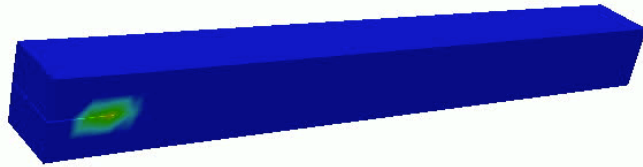
G. Wimmer and H.E. Pettermann. A semi-analytical model for the simulation of delamination in laminated composites. Composites Science & Technology, 68(12):2332 – 2339, 2008.

Isogeometric cohesive elements: 3D example with shells

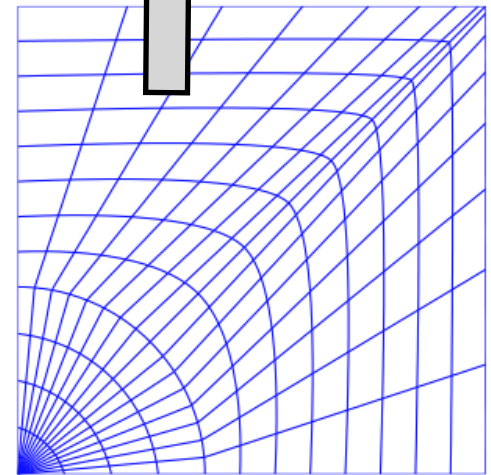
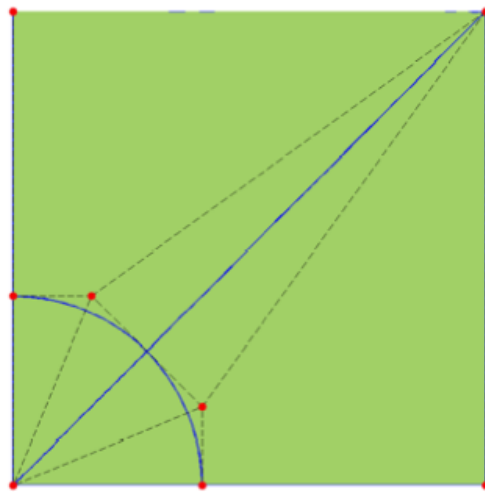


- Rotation free B-splines shell elements (Kiendl et al. CMAME)
- Two shells, one for each lamina
- Bivariate B-splines cohesive interface elements in between

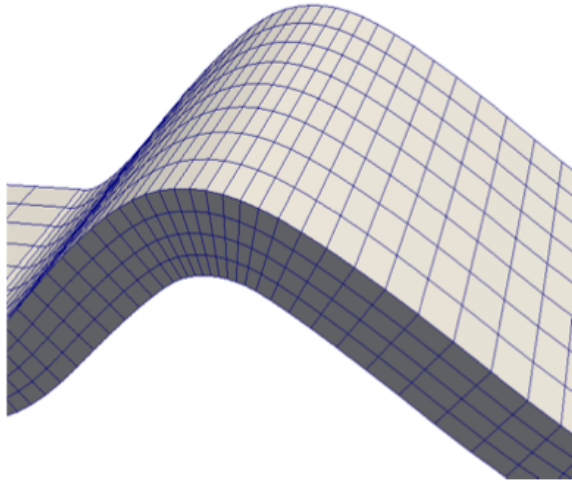
Isogeometric cohesive elements: 3D examples



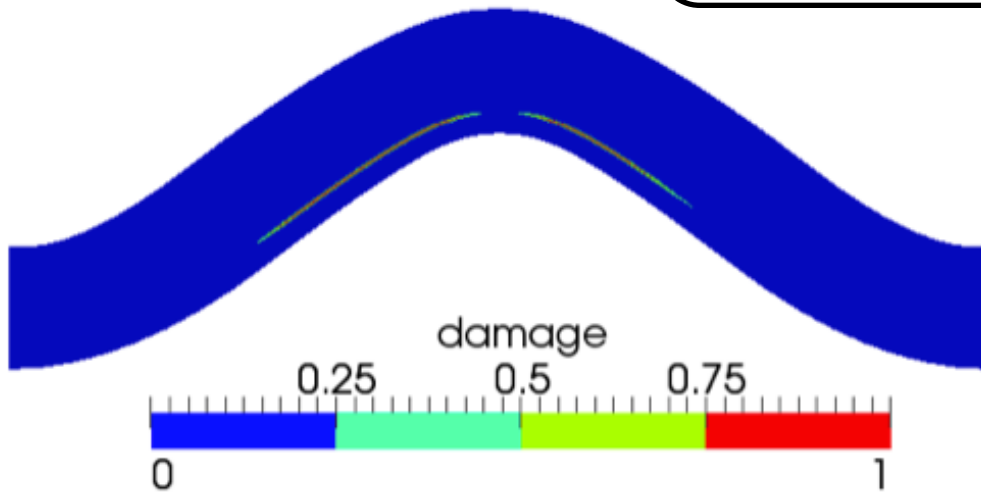
- cohesive elements for 3D meshes the same as 2D
- large deformations
- suitable: delamination buckling analysis



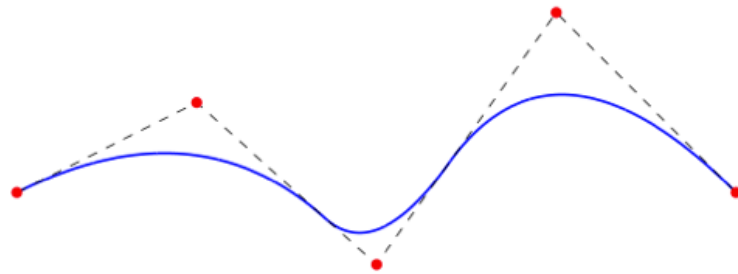
Isogeometric cohesive elements



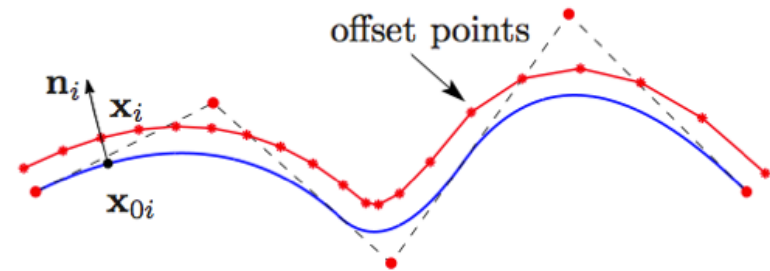
- singly curved thick-wall laminates
- geometry/displacements: NURBS
- trivariate NURBS from NURBS surface
- cohesive surface interface elements
- compression test



Curve offsetting

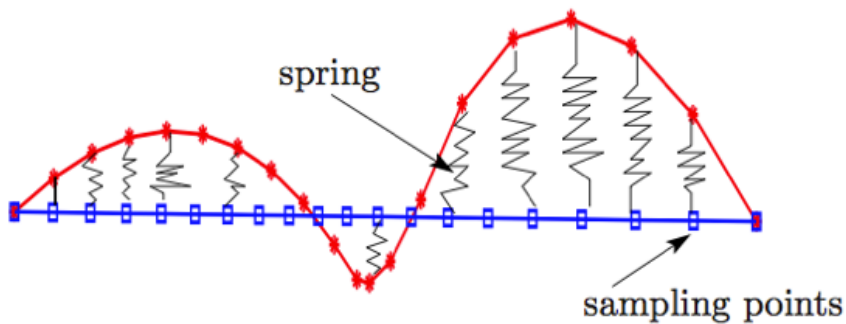


(a)

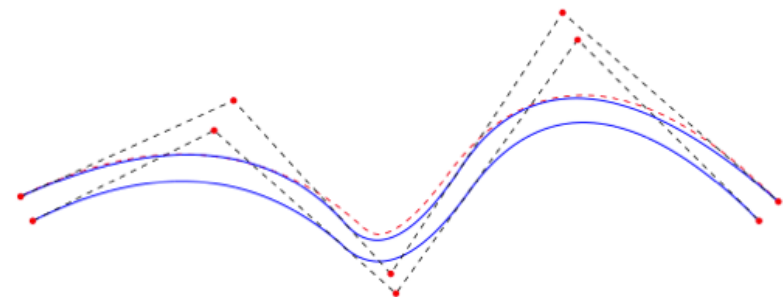


(b)

α



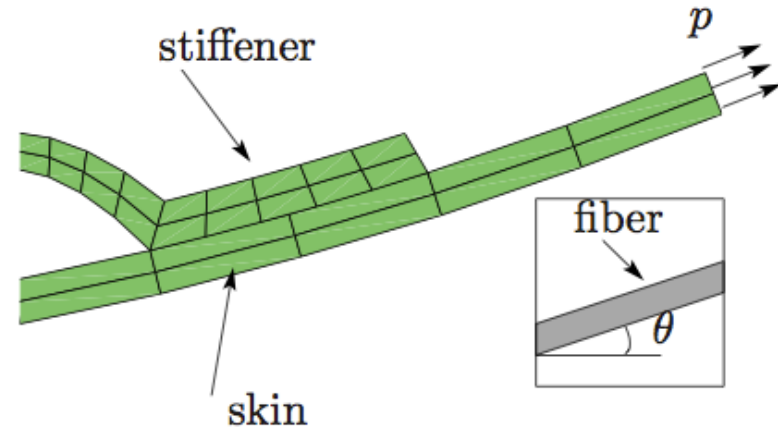
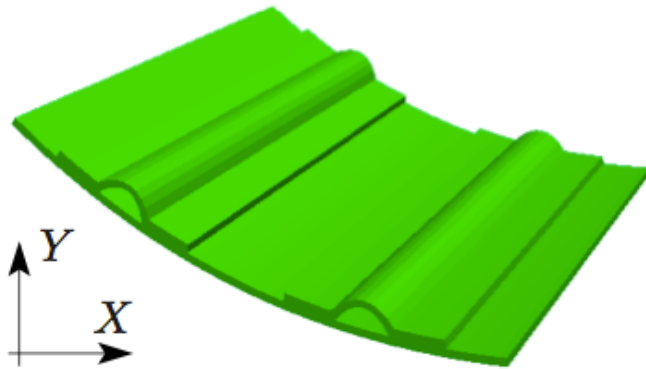
(c)



(d)

(*)V. P. Nguyen, P. Kerfriden, S.P.A. Bordas, and T. Rabczuk. An integrated design-analysis framework for three dimensional composite panels. Computer Aided Design, 2013. submitted.

Curve offsetting



$$E(\mathbf{B}) = \frac{1}{2} \sum_i^m k_s u_i(\mathbf{B})^2 \rightarrow \min$$

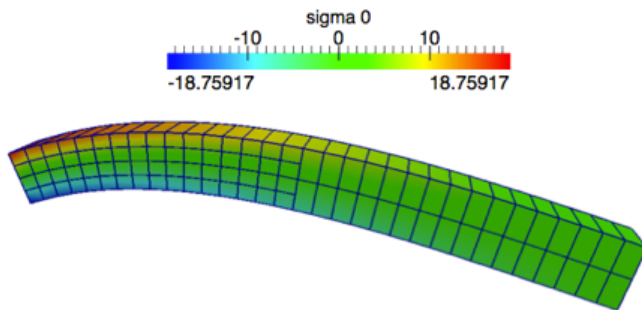
$$\mathbf{B}^{(1)} = \mathbf{B}^{(0)} - \gamma^{(0)} \nabla E^{(0)}$$

$$\gamma^{(0)} = \arg \min_{\gamma^{(0)}} E(\mathbf{B}^{(0)} - \gamma^{(0)} \nabla E^{(0)})$$

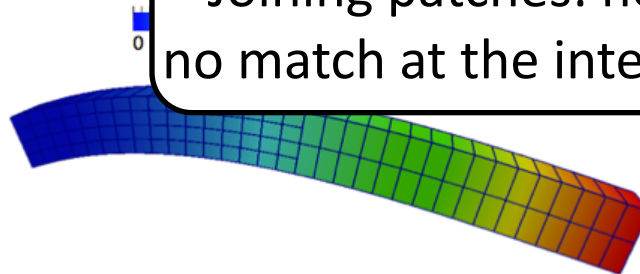
gradient decent method
with line search

Multi patch NURBS

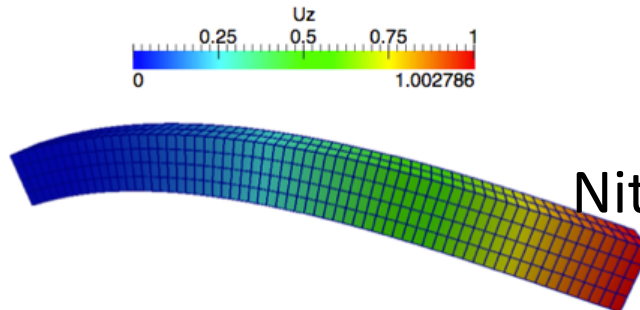
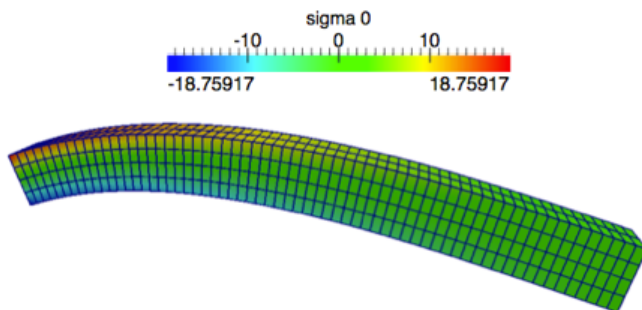
- Tensor-product: 4-sided shape
- Complex geom: multi-patch
- Each patch: its own parametrisation
- Joining patches: not trivial if there is no match at the interface



(a) Nitsche, stress



(b) Nitsche, displacement



Nitsche's method



Concluding remarks

For composite laminates modeling, NURBS IGA offers

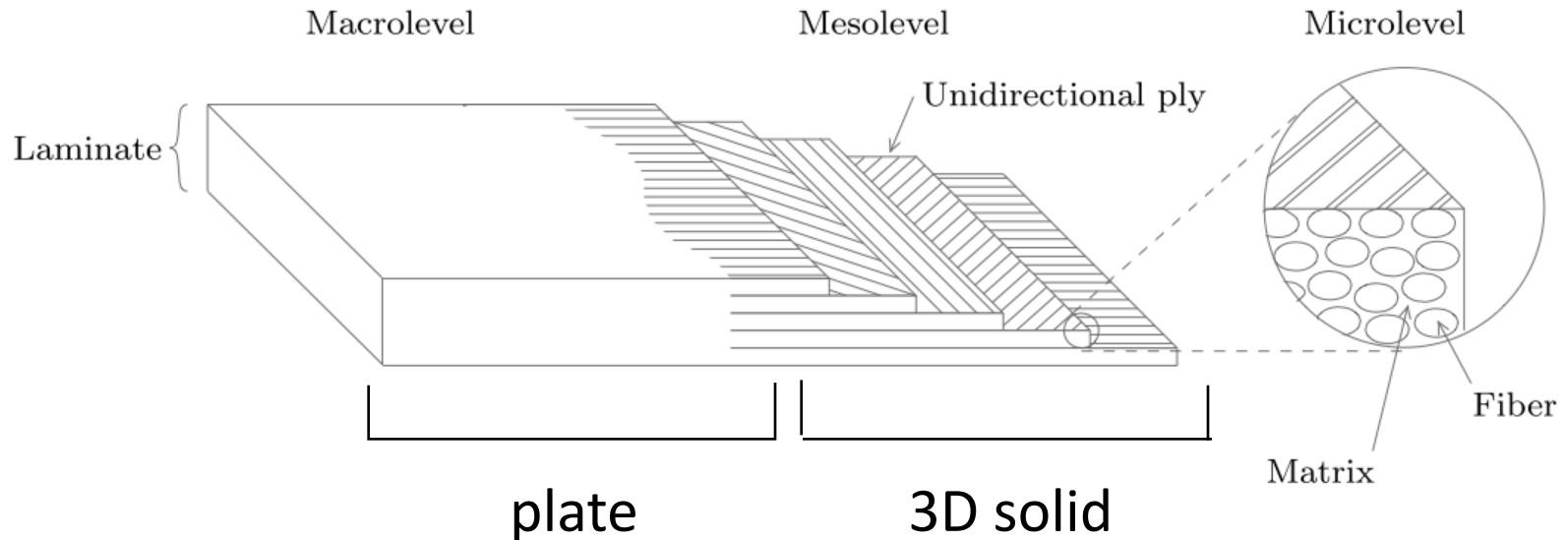
- fast pre-processing: automatic generation of interface el.
- link to CAD data: ideal for design-analysis cycles
- high order NURBS elements: highly accurate derivative fields
- less expensive than low order Lagrange elements
- geometry: exactly represented

- Tensor-product: no local refinement



- T-splines: complex algorithms
- Hierarchical B-splines
- Discontinuous Galerkin methods (NURBS)

On-going and future work



- Multi model coupling: plate (macrolevel) and refined 3D continuum models (mesolevel)
- Macrolevel: through-thickness homogenisation can be used
- Conforming coupling: coupling via an interface
- Non-conforming coupling: 3D model placed anywhere on a

Multi model coupling with Nitsche's method

$$\int_{\Omega} \delta \boldsymbol{\epsilon}^T \boldsymbol{\sigma} d\Omega - \int_{\Gamma_*} [[\delta \mathbf{u}]]^T \mathbf{n} \{\boldsymbol{\sigma}\} d\Gamma - \int_{\Gamma_*} \{\delta \boldsymbol{\sigma}\}^T \mathbf{n}^T [[\mathbf{u}]] d\Gamma + \int_{\Gamma_*} \alpha [[\delta \mathbf{u}]]^T [[\mathbf{u}]] d\Gamma = \int_{\Gamma_t} \delta \mathbf{u}^T \bar{\mathbf{t}} d\Gamma$$

